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HIGH-LOADING, 1800 FT/SEC TIP SPEED TRANSONIC COMPRESSOR FAN STAGE I. AERODYNAMIC AND MECHANICAL DESIGN

by

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16. Abstract				
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HIGH-LOADING, 1800 FT/SEC TIP SPEED TRANSONIC COMPRESSOR FAN STAGE I - AERODYNAMIC AND MECHANICAL DESIGN

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I. SUMMARY

A highly loaded, high tip speed, single stage compressor has been designed under Contract NAS3-13493. This report presents the details of the aerodynamic and mechanical design. The purpose of the program is to determine the feasibility of using high tip speeds and high aerodynamic loadings to obtain high pressure ratios at acceptable levels of efficiency. The rotor has been designed for precompression (external compression) where inlet Mach numbers are high. No inlet guide vane or preswirl was used. The stator vanes were designed for zero exit swirl.

The aerodynamic design was based on the approximate parameters specified in the contract except that the specific flow had to be lowered from 42 lbm/ft²-sec (205 kg/m²-sec) to 38.7 lbm/ft²-sec (188.9 kg/m²-sec) in order to prevent meridional Mach number choking. The rotor blade was designed for a constant spanwise pressure ratio of 2.34:1. Blade losses for the rotor were estimated using a loss model in which shock and profile losses were considered separately. Shock loss estimates were based on relative inlet Mach number and airfoil shape; profile losses were estimated using a correlation of loss parameter versus diffusion factor and precent span. Stator losses were estimated using only a loss parameter correlation.

The rotor was designed using modified multiple-circular-arc blade sections from the hub to 32% span and precompression blade sections from 37% span to the blade tip, with a transition region in between. The purpose of the precompression sections is to reduce the strength of the main passage shock. The rotor was designed using a part-span shroud at 65% span to avoid flutter. The stator was designed with multiple-circular-arc airfoils which approached double-circular-arc geometry toward the outer portion of the span. The design parameters are summarized in Table I.

The stator leading edge was located close behind the rotor trailing edge. The calculated absolute stator inlet Mach number at the hub was 0.892.

Calculated centrifugal stress, gas bending stress, and untwist stress are within the capabilities of the AMS 4972A titanium alloy to be used for the rotor blades. Vibratory blade stresses should be low since the 1E, 2E, and 3E excitations for the first mode are outside of the rig operating range, and the higher order excitations are not expected to occur at low speeds. Although three rotor critical speeds were calculated within the rig operating range, no vibrational problems are anticipated since calculated deflections are within acceptable limits.

TABLE I

DESIGN PARAMETERS

Corrected Speed, rpm	12,464
Rotor Tip Speed, ft/sec (m/sec)	1800 (548.6)
Corrected Flow, lbm/sec (kg/sec)	173.8 (78.8)
Corrected Weight Flow Per Annulus Area,	38.7 (188.9)
1bm/ft^2 -sec (kg/m ² -sec)	
Rotor Pressure Ratio	2.34:1
Stage Pressure Ratio	2.285:1
Rotor Adiabatic Efficiency, Percent	86.8
Stage Adiabatic Efficiency, Percent	84.0
Tip Diameter, Inches (meters)	33.1 (0.84)
Hub/Tip Ratio at Rotor Inlet	0.5
Rotor Tip Solidity	1.635
Rotor Aspect Ratio	2.87
Stator Hub Solidity	2.20
Stator Aspect Ratio	2.22
Stator Exit Flow Angle at all Radii, Degrees	0

II. INTRODUCTION

Future aircraft powerplants will require lightweight highly loaded compressors which are efficient over a wide range of operation. Pressure ratio per stage can be increased considerably above current levels by increasing rotor wheel speed and blade loadings. However, careful consideration must be given to blade element design in order to avoid severe aerodynamic losses. These losses result from in-passage shocks at high Mach numbers, boundary layer growth due to shock impingement, and high blade loadings. Recent tests, Reference 1*, of a highly

^{*}See page 100 for list of References

loaded, 1600 ft/sec (487.7 m/sec), transonic compressor have been very successful; this rotor produces a pressure ratio of 2:1 at an adiabatic efficiency of 89%. Because of this, the design of a higher tip speed, higher pressure ratio rotor was undertaken. Precompression has been incorporated into the blade design in order to maintain a high level of efficiency. The precompression blade design is based on the external compression principle, described in Reference 2.

This report presents the aerodynamic and mechanical design of a single stage fan with a tip speed of 1800 ft/sec (548.6 m/sec) and an inlet hub/tip ratio of 0.5. The design flow per inlet annulus area is 38.7 lbm/ft²-sec (188.9 kg/m²-sec). No inlet swirl is used and the design stator exit swirl is zero. The design stage pressure ratio is 2.285:1, and the predicted stage adiabatic efficiency is 84.0%.

III. FLOWPATH AND VECTOR DIAGRAM DESIGN

A basic stage configuration consisting of no inlet guide vanes, hub/tip ratio of 0.5 at the rotor leading edge, and aspect ratios of 2.87 and 2.22 for the rotor and stator was specified by contract.

The flowpath design evolved from a series of iterations. All values presented are from the final iteration except where otherwise noted. The iteration was started using a reasonable flowpath shape, estimated flow blockages, and estimated efficiency profiles. The axisymmetric streamline analysis calculation outlined in Appendix 1 was used to obtain velocity vectors and flow conditions in the flowpath. This information, together with assumed rotor and stator solidities, was used to design rotor and stator blade elements. Adjustments were made to the flowpath shape and blade solidities to control velocities and loadings, and efficiencies were reestimated using these new loadings and aerodynamic conditions. Blade sections were defined after each iteration to determine the location of blade leading and trailing edges for use in the streamline calculation. The final flowpath design was compatible with blade element designs and the mechanical design.

Rotor losses were calculated as the sum of profile losses and shock losses. The profile losses were calculated using a correlation of loss parameter, $\frac{\omega_p \cos \beta'_2}{2\sigma}$, versus diffusion factor and

percent span. Shock losses were calculated for each particular blade element as described in the Rotor Blade Design section. Different shock loss models were used for the multiple-circular-arc sections (hub to 32 percent span) and the precompression blade sections (37 to 100 percent span). The difference in calculated shock losses for the two airfoil types made it necessary to fair the estimated shock losses in the region from 17 to 54 percent span from the hub. Figure 1 shows the radial distributions of calculated shock total pressure recovery for the MCA and precompression shock models and the faired region between the two models. Figure 2 shows the final radial profile of estimated shock loss coefficients calculated from this pressure recovery curve and the total loss coefficient for the rotor blades.

Stator losses were calculated using a correlation of loss parameter versus diffusion factor and percent span. Figure 3 shows the final radial distribution of stator loss coefficient.

Blockages were included in the aerodynamic design to account for boundary layer growth on the casing walls and for the presence of the rotor part-span shroud at 65 percent span from the hub. Flow blockage due to casing boundary layers was estimated based on data from Pratt & Whitney Aircraft's research programs. To account for the presence of the part-span shroud, a blockage equal to the percent of total annulus area occupied by the shroud was applied locally at the exit of the rotor and to the inlet of the stator; one-half this shroud blockage was applied locally to the rotor inlet and stator exit stations. A constant radial value of blockage due to casing boundary layers was applied at these stations in addition to the part-span shroud blockages. The spanwise blockage factor distributions, in the vicinity of the part-span shroud, are shown in Figure 4. These distributions of flow blockage were used in the streamline analysis program. Table II lists these blockages in terms of radial averages.

TABLE II
FLOW BLOCKAGES (PERCENT OF TOTAL FLOW AREA)

Blade Row and Station Number (Figure 6)	Casing Boundary Layer Blockage	Part-Span Shroud Blockage	Total Blockage
Rotor Leading Edge (1)	2.0	0.6	2.6
Rotor Trailing Edge (2)	3.0	1.2	4.2
Stator Leading Edge (3)	3.0	1.2	4.2
Stator Trailing Edge (4)	4.0	0.6	4.6

The flowpath, including stage inlet and exit ducting, is shown in Figure 5. Figure 6 shows the rotor and stator portion of the flowpath in more detail. A highly convergent flowpath was necessary to keep end wall loadings of both rotor and stator within acceptable limits. High blade solidities were selected to control loadings. Figure 7 shows the spanwise distribution of rotor and stator solidities.

The large convergence together with relatively high aspect ratios gave wall slopes which caused the meridional velocity at the rotor leading edge to accelerate in the mid-span region and decelerate near the walls. Bow wave losses and blockages at the rotor leading edges further increased the meridional velocity. For the desired specific flow of 42 lbm/sec-ft² (205 kg/m²-sec) the meridional Mach number at the center of the span became so high that leading edge blockages caused the flow to choke before it could enter the blade channel. Mid-span choke problems caused by abrupt flowpath convergence are discussed in Reference 3. In order to avoid choking, the flowpath at the rotor leading edge plane was designed with large radii of curvature, and the specific flow was lowered to 38.7 lbm/ft²-sec (188.9 kg/m²-sec). A comparison of the inlet meridional Mach number profiles for 42 and 38.7 lbm/ft²-sec (205 and 188.9 kg/m²-sec) is shown in Figure 8. Calculation of boundary layer shape factors along the inlet walls using the method of Reference 4 indicated that the decelerating meridional velocity would not cause the end wall boundary layer to separate.

Figure 9 shows the rotor and stator inlet and exit meridional velocity profiles. The average velocity at the stator exit was 680 ft/sec (207 m/sec). Inlet Mach numbers for the rotor and stator are shown in Figure 10. The rotor inlet relative Mach number is supersonic from 9.5 percent span to the tip. The rotor exit relative Mach number is subsonic throughout the span. The stator inlet Mach numbers vary from approximately 0.9 at the hub to approximately 0.7 at the tip. By minimizing the gap between the stator leading edge and the rotor trailing edge, the absolute stator inlet Mach numbers and loadings were minimized.

Figure 11 shows the rotor inlet and exit relative air angles along with the spanwise stator inlet angles. The stator discharges axially, $\beta'_3 = 0^\circ$. An upturn in the rotor exit relative air angle, β'_2 , occurs between 90 and 100 percent span in the boundary layer region due to the high tip losses estimated for the rotor. Difficulties were encountered in selecting blade sections near the tip because of this abrupt rise in β'_2 , and made it necessary to fair a continuous curve of β'_2 versus span through the rotor tip region. The faired β'_2 curve is shown in Figure 11.

Rotor and stator diffusion factors are shown in Figure 12. Hub D-factors for the rotor and stator are quite high even though the design has high solidity rotors and stators. Stall characteristics of this stage are expected to be strongly influenced by these high hub loadings.

Figure 13 shows the spanwise distribution of rotor and stage adiabatic efficiencies. The mass-averaged efficiencies are 86.8 percent for the rotor and 84.0 percent for the stage.

Spanwise distributions of local specific flow, $(W/A)_{local} = \rho V_z$, at the rotor and stator leading and trailing edges are shown in Figure 14. The decrease in specific flow at the rotor tip from leading to trailing edges and the large increase across the stator tip in relationship to the average increase across the stator, are caused by the expected rotor tip losses. High rotor work in this high loss area gives a large swirl velocity and a low axial velocity so that tip stream tubes expand through the rotor. When the swirl is removed by the stator, the outer stream tubes contract. The stream tube expansions and contractions at the tip are beyond the experience on which the blade design system is based and some discretion in selecting airfoils in the outer span was required; this will be discussed further in Sections IV and V.

The design velocity vector data calculated along streamlines at the rotor and stator leading and trailing edges are tabulated in Appendix 2, Aerodynamic Summary.

IV. ROTOR BLADE DESIGN

The rotor blade was designed to produce a total pressure ratio of 2.34:1 at a tip speed of 1800 feet per second (548.6 m/sec). There are 38 rotor blades with an aspect ratio of 2.87 (based on average blade length and axial projected chord at the hub) and a tip solidity of 1.635.

The rotor blade consists of multiple-circular-arc (MCA) airfoil sections from hub to 32% span and precompression (PC) sections from 37% span to the tip, with a transition region in between. Transition blade sections were necessary to provide reasonably smooth airfoil shapes

between the MCA and precompression airfoil sections. Precompression airfoils were chosen to reduce shock losses wherever possible, i.e., wherever the relative inlet Mach number and blade leading edge wedge angle allowed an attached oblique shock. Airfoil sections were designed on conical surfaces approximating stream surfaces of revolution.

A preliminary parametric study of blade mechanical properties was made assuming MCA airfoils at all spanwise locations since the MCA sections are more easily specified than PC sections. The more complicated PC airfoil design was begun after general mechanical design criteria were established. Essentially the same mechanical properties were maintained in the blade with PC as MCA airfoils by adjusting the PC section at each spanwise position to obtain the same cross section area.

Multiple-Circular-Arc Design

MCA sections extend from the root to 32% span. True multiple-circular-arc airfoils were used in the design procedure. These airfoils were subsequently thinned to provide more flow area, and these thinned MCA airfoils will be referred to as "modified" MCA airfoil sections in this report.

Airfoil sections were defined by specifying total and front chord, total and front camber angle, maximum thickness and its location, and leading and trailing edge radii, as shown by Figure 15a. The front chord was selected to provide a transition point just forward of an assumed normal shock impingement point on the suction surfaces.

For the MCA blade sections, a normal shock is assumed at the first covered section of the blade passage as shown in Figure 15b. A Mach number upstream of the assumed normal shock is calculated by using the equations of continuity and conservation of angular momentum. This calculation accounts for stream tube contraction and radius change. Next, the critical area ratio (A/A^*) resulting from the Mach number calculation is adjusted to account for blade blockage by multiplying the A/A^* by the ratio of blade entrance channel width to $(s \cos \beta'_1)$. The resulting A/A^* determines the upstream shock Mach number. The shock total pressure recovery (shown in Figure 1) was than computed based upon the assumption of a normal shock at this upstream Mach number.

Maximum thickness-to-chord ratio at the root is 0.077 which gives calculated combined steady state stresses, including untwist stresses, within the allowable limits of AMS 4972A titanium alloy selected for blade material. The spanwise distribution of maximum thickness-to-chord ratio is shown in Figure 16. Total and front chords for the modified MCA rotor blade are shown in Figure 17.

Incidence angle to the suction surface at the leading edge versus span is shown in Figure 18. Two incidence criteria were used: 1) subsonic and transonic MCA, and 2) supersonic MCA.

Rotor incidences for the subsonic and transonic blade sections were set based on measured minimum loss incidence angles in Reference 1. Rotor incidence for the supersonic MCA

portion of the blade was selected as +1.5 degrees to the suction surface at a location half-way between the leading edge and the emanation point of the first captured Mach wave. This incidence selection is based on experience and accounts for bow wave shock loss, blade leading edge blockage, and development of the suction surface boundary layer.

The ratio of minimum blade channel flow area to critical area, A/A*, was chosen as 1.02 to prevent choking. Distributions of A/A* through five blade channels between 0 and 30.3 percent span are shown in Figure 19. Area distributions are along conical surfaces. The critical A/A* ratio was calculated knowing the channel width, shock loss, and the assumptions of a linear specific flow variation from the leading edge to the trailing edge and a linear profile loss distribution from the blade channel entrance to the trailing edge. Front camber was the main parameter used to control blade channel width. An improved A/A* calculation technique accounting for streamline radius ratio was developed after completing the rotor design. This calculation showed that the channel areas for sections from 9 to 32 percent of the rotor span were too small. Additional flow area was obtained by locally thinning the standard MCA airfoils as shown in Figure 20. The A/A* distributions for this modification are included in Figure 19. Blade total and front camber angle distributions are shown in Figure 21.

Deviation was estimated using Carter's rule plus a correction based on a correlation of test data from Reference 1. The design deviation and Carter's rule deviations for the MCA blade sections are shown versus span on Figure 22.

Precompression Blade Design

For the outer radii, a low shock-loss precompression blade section was used because of the high rotor tip speed and high inlet relative Mach number associated with this rotor. Figure 23 is a schematic of the precompression airfoil. A general description of the precompression blade design is presented in the following paragraphs.

The precompression design model assumes that the shock across the channel entrance must be oblique and attached to the leading edge of the airfoil. Flow conditions upstream of the first captured Mach line are adjusted to account for the bow shock system which propagates upstream of the rotor inlet plane. The concave surface (BC in Figure 23) is the precompression ramp. The curvature of this ramp generates a series of compression waves which diffuse the supersonic flow. The wave system is designed to coalesce near A', slightly downstream of the rotor inlet plane. The precompression wave system lowers the Mach number of the flow across the passage entrance, reducing the total pressure loss associated with the oblique shock (A'-D on Figure 23). The flow deflection across this oblique shock is equal to the effective leading edge wedge angle (blade wedge angle plus boundary layer displacement) plus the precompression angle.

The shock A'-D is constructed in increments across the gap to account for gapwise changes in flow conditions upstream of the shock. Shock losses are calculated for each increment and are mass averaged across the gap to obtain the oblique shock loss of the blade element. Channel flow downstream of the oblique shock is subsonic, and turning and stream tube area are made compatible with exit aerodynamic conditions.

Development of the blade suction surface AB beings by aligning the aerodynamic surface (blade surface plus boundary layer displacement) to a constant angular momentum streamline from the leading edge to the first captured Mach wave. Suction surface curvature in segment CD is designed to adjust the supersonic flow upstream of the shock to be compatible with the shock deflection and subsonic flow condition downstream of the shock. Iterations were made on surface shape until compatibility was achieved, accounting for effect of radius change and stream tube convergence (or divergence).

The suction surface immediately behind the shock impingement, D, is aligned with the flow direction downstream of the shock. The surface is rounded at D to allow for boundary layer thickness changes in the region of the shock impingement. Channel area is blended from the value at D to the area determined by the core flow at the channel exit. A cosine variation of stream tube area determines the locus of points which define the suction surface (DG). The cosine variation equations are given in Appendix 3.

The pressure surface segment AE follows a free streamline downstream of the oblique shock. Segment FG of the pressure surfaces is designed to guide the flow to the desired exit angle. Segment EF blends smoothly between AE and FG. The chordwise locations of the pressure surface points, E and F, are tabulated in Appendix 4.

The resulting mean-line incidence and deviation angles for the entire blade are shown in Figures 24 and 25, respectively. The mean-line metal angles used in the calculation of these figures are average pressure and suction surface metal angles at the leading and trailing edges of the developed blade sections.

Precompression blade sections were designed for a radial distribution of cross-sectional area determined from the preliminary MCA design. The main purpose of this requirement was to provide good mechanical properties and blending of the MCA and PC blades. The blade spanwise cross-sectional areas are shown in Figure 26; this curve shows the areas for the modified MCA blade (dashed line) as well as the true MCA areas (solid line). The PC airfoil cross-sectional areas, which are shown in Figure 26, were obtained primarily by controlling the blade leading edge wedge angle and leading edge radius (Figure 27). The leading edge wedge angle and the location of the pressure surface point, E, (Figure 23) were chosen to avoid local chordwise narrowing of blade elements which could result in concentrations of vibrational stress.

Channel area between the blades was calculated from the aerodynamic blade surface contours and estimated stream tube height from the streamline analysis. This area was then increased to account for boundary layer displacement thicknesses which were determined using the boundary layer calculation of Reference 4. In this calculation, the abrupt pressure rise, caused by the passage shock wave, is spread linearly across the shock impingement - from five boundary layer thicknesses upstream of the shock impingement point to the trailing edge.

A minimum critical area ratio (A/A^*) was calculated for the precompression sections based upon channel width between blades, bow wave losses, shock losses, streamline radius changes, and specific flow. This minimum A/A^* ratio occurs just downstream of the oblique shock

(A'D). The minimum A/A* ratios for the precompression blade sections are plotted spanwise in Figure 28 for three types of assumed shock losses. Curve A of Figure 28 is computed on the basis of the design shock loss, i.e., the oblique shock of Figure 23; for this design case, the (A/A*)_{min.} ranged from 1.02 to 1.065, with a drop-off at the tip region due to high end wall losses. Curve B in Figure 28 is the minimum A/A* distributions based on a normal shock at the channel entrance Mach number (e.g., depicted in Figure 15b), and curve C is based on a normal shock at the inlet relative Mach number, M₁'. Figure 28 shows that the flow will start if the blades are subjected to normal shock losses based on a channel entrance Mach number, but most of the flow in the outer 40 percent span will not start if these losses are based on M₁'; i.e., the Kantrowitz-Donaldson criterion (Reference 5). It is thought, however, that the normal shock at M₁' is an extreme condition which will not occur due to the presence of the diffusing wave system associated with the precompression ramp. It is more reasonable to expect that in starting, the rotor passage will encounter a normal shock at a precompressed Mach number, so that the starting $(A/A^*)_{min}$ distribution may be similar to curve B. It is believed that adequate flow area is provided for design entrance flow conditions to be established in the precompression blade sections under these conditions.

Blade chord, leading and trailing edge metal angles, and precompression ramp angles are shown in Figures 29, 30, and 31 for conical surfaces on streamlines. Rotor blade geometry is tabulated in Appendix 4, Table VII. Also included in Appendix 4 is a graphical description of an airfoil on the unwrapped conical surface (Figure 50). This figure is used in conjunction with Table VII. Rotor tip geometry was selected using the previously discussed faired β'_2 curve of Figure 11. For manufacturing purposes, the airfoil sections were defined on planes normal to a radial (stacking) line. The resultant blade coordinates are presented in Appendix 5, Table IX. Airfoil coordinate labels used in Table IX are graphically defined in Figure 51 (included in Appendix 5).

Transition

The region of the blade from 32 to 37% span provides transition from the modified MCA to PC sections. The last MCA conical section is at 32% span and the first PC section is at 37% span. The blade section stacking program which defines blade surfaces by a parabolic curve fit between conical input sections was used to determine transition blade sections. For manufacturing purposes, the airfoils in the 32-37 percent span region were defined on planes normal to a radial line.

V. STATOR VANE DESIGN

The stator has multiple-circular-arc airfoil sections (Figure 15a) with sections at the outer span approaching double-circular-arc (DCA). Airfoil sections were designed on conical surfaces approximating streamsurfaces of revolution.

The aerodynamic chord length tapers linearly from 2.48 inches (0.063 m) at the hub to 2.6 inches (0.066m) at the tip. The stator has 60 vanes resulting in a hub solidity of 2.2. Aspect ratio is 2.22 based on average blade length and axially projected chord at the hub. Maximum thickness-to-chord ratio is 0.07 at the tip tapering learly to 0.05 at the hub. This stator vane thickness distribution was selected to provide mechanical integrity and low blade element loss.

Front chord length (c_f in Figure 15) for hub sections was selected to set the transition point just forward of the intersection on the suction surface of a line drawn from the leading edge of an adjacent blade and normal to the flow. Airfoil sections near the outer case have front chords approximately one-half the total chord and front cambers approximately one-half the total camber. Therefore, these sections are essentially DCA airfoils. Figure 32 shows the spanwise distributions of $\frac{c \sin \phi_f/2}{c_f \sin \phi/2}$ which is the average airfoil mean-line radius of

curvature divided by the front section-mean-line radius of curvature. This parameter equals 1.0 for DCA airfoils and becomes smaller as the front section is uncambered relative to a DCA airfoil. Figure 33 shows the chordwise location of airfoil maximum thickness versus span.

Incidence angle was set at zero degrees to the suction surface in the near-sonic flow region at the hub, blending into minimum loss incidence angles for double-circular-arc sections near the tip (Figure 34). Camber distribution was used to control throat area in the channel between adjacent vanes. The optimum ratio of capture area to throat area, which provides minimum loss (Reference 6) varied from 0.97 at the hub to 0.985 at the stator tip. This capture-to-throat area parameter was used to set throat areas. Figure 35 shows the axial distributions of A/A^* in channels between stator airfoil sections.

Incidence angle and front camber did not control throat areas in the outer 15 percent span where the extremely low ratio of inlet to exit specific flow, $(\rho V_z)_3/(\rho V_z)_4$, caused the minimum throat area to occur at the channel exit (Figure 35, 100 percent span). Minimum channel flow area is lower than the optimum indicated by Reference 6, although A/A* is adequate to prevent choking. Incidence angle selection was complicated by the unusually low specific flow ratio, $(\rho V_z)_3/(\rho V_z)_4$, and high inlet air angle which fell outside the range of available stator blade-element data. Pratt & Whitney Aircraft cascade data for geometrically similar DCA cascades but with specific flow ratios closer to 1.0 were considered most applicable and were used to select stator tip incidence angles.

Stator deviation angles were determined using Carter's rule plus an adjustment based on data from References 1 and 7. The spanwise distribution of Carter's rule and design deviation angles are given in Figure 36.

Figure 37 presents mean-camber-line metal angles versus span, and Figure 38 presents front and total camber angles. All angles in these figures are measured on conical surfaces on which the airfoils were designed. Stator vane geometry on conical surfaces is summarized in Appendix 4, Table VIII. Also included in Appendix 4 is a graphical description of an airfoil on the unwrapped conical surface (Figure 50). This figure is used in conjunction with Table VIII.

For manufacturing purposes, airfoil sections were defined on planes normal to a radial line which passes through the center of gravity of the hub section. Coordinates of these sections are tabulated in Appendix 5, Table X. Airfoil coordinate labels used in Table X are graphically defined in Figure 51 (included in Appendix 5).

VI. STRUCTURAL AND VIBRATION ANALYSIS

The mechanical design included an investigation of the rotor and stator airfoil steady and vibrational stresses and flutter parameters. The natural modes of the rotor and stator systems were calculated and a rotor-frame critical speed analysis was performed to investigate the vibrational characteristics of the fan stage.

The final design is stress limited in the blade attachment to a mechanical speed of 13,650 rpm which is 109.5 percent of the standard day design speed of 12,464 rpm and 106.5 percent of design speed based on a 90°F (305.2°K) rig plenum temperature. All mechanical analyses are based on this 90°F plenum temperature.

A. Rotor Blade and Stator Vane Stresses

Combined centrifugal pull and untwist rotor blade stresses were calculated at 110 percent of design speed and results are shown in Table III along with the allowable stresses for the blade material, AMS 4972A titanium alloy bar stock, based on 200°F (366.3°K) metal temperature. The maximum combined centrifugal pull and untwist stress of 77,700 psi $(536 \times 10^6 \text{ N/m}^2)$ occurs at 13 percent span, near maximum thickness on the concave side of the airfoil. This stress is comparable to stress levels present in experimental and production blades and is well below the allowable stress of 94,000 psi $(648 \times 10^6 \text{ N/m}^2)$. The maximum vibratory stress occurs at 22 percent span near the trailing edge on the concave side of the airfoil. The static stress in this area is only 54,000 psi (372×10^{6}) N/m³). Figure 39 shows these maximum stress locations. A modified Goodman diagram, Figure 40, indicates that, at the maximum steady state stress level of 77,700 psi (536 x $10^6 \,\mathrm{N/m^2}$), the maximum allowable vibratory stress is slightly above 10,000 psi (69 x 10⁶ N/m²). Since no low order resonances are expected in the blade's high speed operating range, the actual vibratory stress levels should be less than the allowable value of 10,000 psi $(60 \times 10^6 \text{ N/m}^2)$ indicated by the Goodman diagram for this point on the blade. During testing, the vibratory stress will also be limited to 10,000 psi (69 x 10⁶ N/m^2).

Gas bending stresses with centrifugal restorations were calculated at design speed for various tangential tilts of the blade, see Figure 41. Airfoil stresses were minimized for the combination of load and no-load conditions. The selected tangential tilt is 0.050 inch (0.00127 m) which results in a maximum tensile bending stress of 7000 psi $(48 \times 10^6 \text{ N/m}^2)$ at 12,464 rpm.

The part-span shroud was designed to insure minimum aerodynamic interference while providing adequate strength under centrifugal loading and bearing forces. The part-span shroud is located at 65 percent of span. A sketch of the part-span shroud is shown in Figure 42. The Z* ratio, defined as shroud section modulus/adjacent airfoil section modulus, has a value of 0.84 which is consistent with successful experience. Pertinent shroud stresses are shown in Table III.

Stator vane (AMS 5613 stainless steel material) gas bending stresses were calculated assuming both fixed and pinned ends. The maximum bending stress of 38,300 psi

 $(264 \times 10^6 \ \text{N/m}^2)$ was well below the allowable of 108,000 psi $(755 \times 10^6 \ \text{N/m}^2)$. Maximum allowable vibratory stress limits for stator vanes are established based upon test experience with similar stator vane and attachment designs. For this stator, the test limit was the same as for the rotor; i.e., $10,000 \ \text{psi}$ $(69 \times 10^6 \ \text{N/m}^2)$. This stress limit is conservative because the stator's steady state stresses are lower than the rotor's steady state stresses, and, like the rotor, there are no critical stator vane resonances in the operating range.

TABLE III

CALCULATED STRESSES FOR ROTOR BLADE, BLADE DISK, AND STATOR VANE

38 Rotor Blades 60 Stator Vanes Disk AMS 4972A (titanium alloy) AMS 5613 (stainless steel) AMS 6415 (low-alloy steel)

Assumed Operating Conditions

N = 110% of design speed, 90°F (304.2°K) plenum temperature, except where noted

 $T = 200^{\circ} F (366.3^{\circ} K)$ metal temperature

Stress	Calculated	Allowable
Blade airfoil max. combined stress	77,700 psi (536 x 10 ⁶ N/m ²)	94,000 psi (648 x 10 ⁶ N/m ²)
Blade airfoil max. root com- bined stress corrected for platform angle (at leading edge)	77,600 psi (536 x 10 ⁶ N/m ²)	94,000 psi (648 x 10 ⁶ N/m ²)
Blade shroud bearing stress	4075 psi (28 x 10 ⁶ N/m ²)	5000 psi (34 x 10 ⁶ N/m ²)
Blade shroud bending stress	64,700 psi (446 x 10 ⁶ N/m ²)	66,000 psi (455 x 10 ⁶ N/m ²)
Blade attachment max. combined stress (106.5% speed)	71,022 psi (490 x 10 ⁶ N/m ²)	71.077 psi (490 x 10 ⁶ N/m ²)
Blade attachment max. bearing stress (106.5% speed)	87,532 psi (604 x 10 ⁶ N/m ²)	84,600 psi (583 x 10 ⁶ N/m ²)
Disk lug max. combined (106.5% speed)	63,633 psi (439 x 10 ⁶ N/m ²)	103,367 psi (713 x 10 ⁶ N/m ²)
Disk lug bearing stress (106.5% speed)	87,532 psi (604 x 10 ⁶ N/m ²)	122,400 psi (844 x 10 ⁶ N/m ²)
Disk average tangential stress	82,950 psi $(572 \times 10^6 \text{ N/m}^2)$	106,000 psi (731 x 10 ⁶ N/m ²)
Disk max. radial stress	74,000 psi (510 x 10 ⁶ N/m ²)	96,000 psi (662 x 10 ⁶ N/m ²)
Vane airfoil max. bending stress	$38,300 \text{ psi}$ (264 x 10^6 N/m^2)	108,000 psi (745 x 10 ⁶ N/m ²)

B. Rotor Blade Attachment and Disk Stresses

Critical speed considerations required that disk rim weight be minimized, necessitating the selection of a dovetail attachment rather than a firtree attachment. The final blade attachment design is stress limited to a mechanical speed of 13,650 rpm which is 109.5 percent of the standard day design speed of 12,464 rpm and 106.5 percent of design speed based on a 90°F (305.2°K) rig inlet temperature. The bearing stress in the blade attachment exceeds the nominal allowable by 3.5 percent. This is acceptable since the allowable limit is conservative and intended for production engines. All disk (AMS 6415 low-alloy steel material) stresses are well below allowable limits. See Table III for a summary of blade attachment and disk stresses.

The airfoil root stress/blades attachment stress is approximately 2.0, insuring that the attachment can withstand vibratory stresses higher than the airfoil can tolerate.

C. Rotor Blade and Stator Vane Resonances

Coupled blade-disk resonances which might be excited in the operating range were avoided by the proper choice of shroud location, shroud angle, blade material, and disk geometry and material. Low order excitation from circumferential distortion or other possible inlet pressure variations will not excite the system because the blade and disk were designed to insure that natural modes for the system would not occur at frequencies close to one, two or three excitations per revolution (1E, 2E, or 3E) during high speed operation. The bladed disk resonance diagram is shown in Figure 43. Performance testing will be avoided at those speeds where higher order resonances (6E, 8E, or 10E) might occur. However, there are no struts or instrumentation in the system to excite these resonances. The first bending mode 3E frequency margin is more than 8 percent at 105 percent of design speed, and more than 5 percent at 110 percent of design speed, which is considered adequate. Blade strain gages will indicate any resonant conditions that exist and test speeds can be adjusted accordingly to avoid operation at these resonant conditions.

The blade tip dynamic stress distribution in the first chordwise bending mode is shown in Figure 44. The maximum stress is six times the stress at maximum blade thickness and occurs at 25 percent chord. However, no tip fatigue problems are expected since the first tip chordwise bending mode occurs in the region of low static stress and is not excited by vane passing order or any 8E or lower frequency excitations as shown in Figure 45. A 10E resonance occurs in the high speed operating range but no evidence of this excitation was found in the stress records of a fan stage using the same inlet case and almost identical instrumentation. Higher order resonances are not expected to cause problems since they occur at lower speeds. The maximum stress location caused by the chordwise bending modes will be determined so that the blade tips can be adequately strain gaged.

The stator vane resonant frequencies were calculated assuming that the vanes were fixed at both ends of the airfoil and then adjusted based on past testing of similarly mounted vanes. The stator resonance diagram is shown in Figure 46. At design speed, the only resonance that exists is 5E for which there is no anticipated source of excitation. Below design speed the 38E vane resonance or other resonances can be avoided by proper selection of test speeds to avoid resonant conditions indicated by vane strain gages. As an additional safeguard against encountering high stresses throughout the operating range, the stator vane inner diameter shank will be supported with a flexible bushing as shown in Figure 47. Damping of stator vane vibration by supporting vanes in a flexible (polyurethane) compound has been successfully demonstrated in an engine program.

D. Rotor Blade and Stator Vane Flutter

To avoid rotor blade flutter a part-span shroud was located at 65 percent span from the hub with a shroud contact angle of 25 degrees. The supersonic torsional flutter parameter $(V/b\omega_t)$ was calculated for the portion of the blade above the part-span shroud. The V and b terms are the relative velocity and semi-chord at 75 percent span for this portion of the blade, and the ω_t term is the torsional frequency of the blade portion above the shroud assuming the blade is fixed at the part-span shroud location. The flutter parameter was 1.02 at design speed and 1.04 at 110 percent of design speed; these values are below typical values where flutter problems occur; therefore, no supersonic torsional tip flutter is expected throughout the high speed running range.

The stator vane bending flutter parameter and torsional flutter parameter were calculated for these vanes; the calculated points fall well into the successful experience (no flutter) area.

E. Critical Speed

A rotor-frame critical speed analysis was performed to determine the vibrational characteristics of this rig. The analysis includes all of the significant structural members and uses the spring rates, masses, and gyroscopic stiffening of the system. The compressor rig spring location and spring rates are shown in Figure 48.

Three critical speeds occur within the rig operating range at 5500, 11,800 and 12,500 rpm. A fourth one occurs at 14,800 rpm which is above the maximum operating speed. The mode shapes at these four speeds are shown in Figure 49. To determine if the vibratory amplitudes of these modes are acceptable, a forced response analysis was performed. This analysis is similar to the critical speed analysis except that an unbalance is simulated and the resultant vibratory deflections calculated. Deflections were calculated at all significant rotor and case locations for an unbalance of one (1) ounce-inch $(72 \times 10^5 \text{ kg-m})$ located at each of five (5) locations: disk, No. 2 bearing, diaphragm coupling, forward spline, and rear spline. Table IV gives a listing of estimated deflections for the four (4) critical speeds, or modes.

TABLE IV

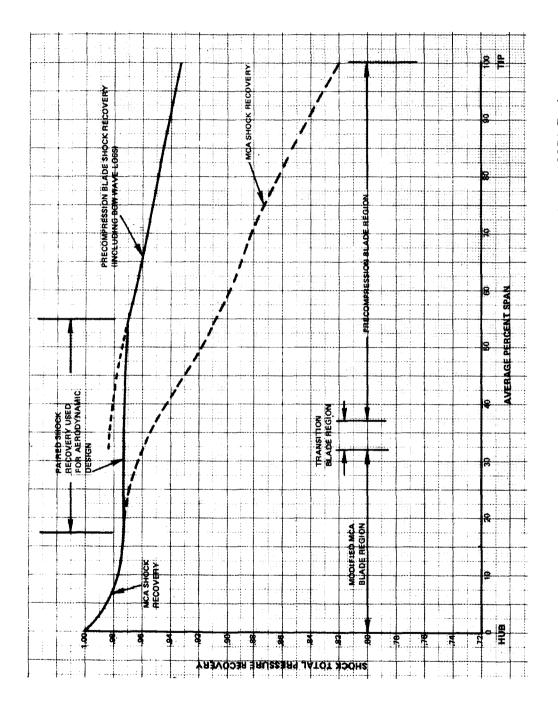
CALCULATED DEFLECTION FOR RIG CASINGS AND SUPPORT MEMBERS

1.0 Ounce-Inch (72 x 10⁻⁵ kg·m) Unbalance* NOTE: 1 Mil = 2.545 x 10⁻⁵ Meters

Unbalance Location		1	Disk		Dit	hragn	Diphragm Coupling	ling		No.	No. 2 Bearing	ing		Forward Spline	d Splii	ન		Rear	Rear Spline	
Mode	1	7	ε	4	-	7	κ	4	-	7	c	4	-	2	ъ	4	-	7	8	4
Response Location																				
Disk	•	12	1		٠	3	•	,	,	7		,	,	7	ı			ъ		•
ID Stator Case		-			-		•	•	-			,	,			,				
OD Case	-	7			11	•			-				-	4						
Bellmouth	7		2		4		•	•	4				7	,		1	-	,		
ID Inlet Fairing		19	19		ю	∞	œ	4	4	8	\$		2	9	9		_	4	4	•
No. 1 Bearing		9	-	٠	•	7		-		-		•	•	5		-		-		
No. 1 Brg. Support	•	S	-		٠	1	•	-		-		•	•	-		1	-			_
No. 2 Bearing	•	7			1	1	-	က	,	1		2	•	-	-	4				c
No. 2 Brg. Support	-	-			-	•			-				•							•
No. 3 Bearing	•	7	•		•	•		9	•			8		-	-	9		•		2
No. 3 Brg. Support	•	-			•	•	•	7	•			-				3				7
No. 4 Bearing		•		•	•	•	٠	•				•	•							
No. 4 Brg. Support	,	•		ı	٠	•					•		•	•	•	ı				•
Inlet Struts		•	i		•	•	•					•	•	•		•				•
Exit Struts		-	•	•	-	•	•	•			•	•	•	•		•				
NOTE: Mode 1:	550	5500 RPM	7		Mode 2:	2:	118	11800 RPM			Mod	Mode 3:	12500	12500 RPM		Mode 4:		1480	14800 RPM	_

The amplitudes in this table are approximate values and are estimated to be maximum values.

The 11,800 rpm mode was found to be the most sensitive to unbalance. The largest estimated deflections (for 1 ounce-inch unbalance) were 0.019 inches (0.0005 m) at the inlet fairing and 0.012 inches (0.0003 m) at the disk. Deflections at the inlet fairing are no threat to rig safety. The rotor assembly (consisting of disk, blades, forward and rear shaft assemblies, and diaphragm coupling) will be balanced to better than 0.05 oz-in $(36 \times 10^{-5} \text{ kg-m})$ unbalance resulting in a maximum disk deflection of 0.005 inches (0.00013 m). This is considered acceptable since the minimum blade tip clearance is 0.040 inches (0.0010 m) and the rig is adequately instrumented to detect and avoid critical speed regions. Also, actual sensitivity to unbalance may be considerably less since conservative values of damping coefficients were used in the analysis.



Spanwise Shock Recovery for Rotor Precompression and MCA Designs Figure 1

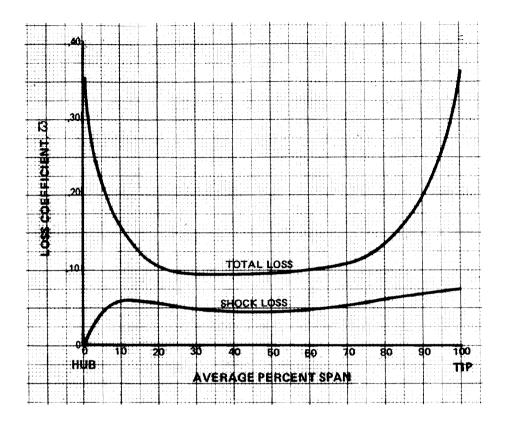


Figure 2 Rotor Blade Total and Shock Loss Coefficients (Design)

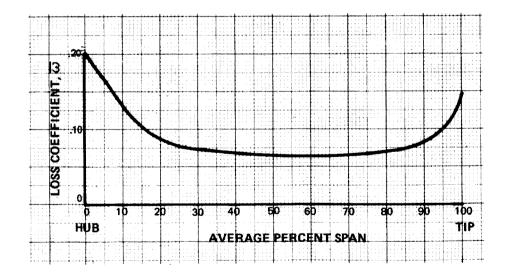


Figure 3 Stator Vane Loss Coefficient (Design)

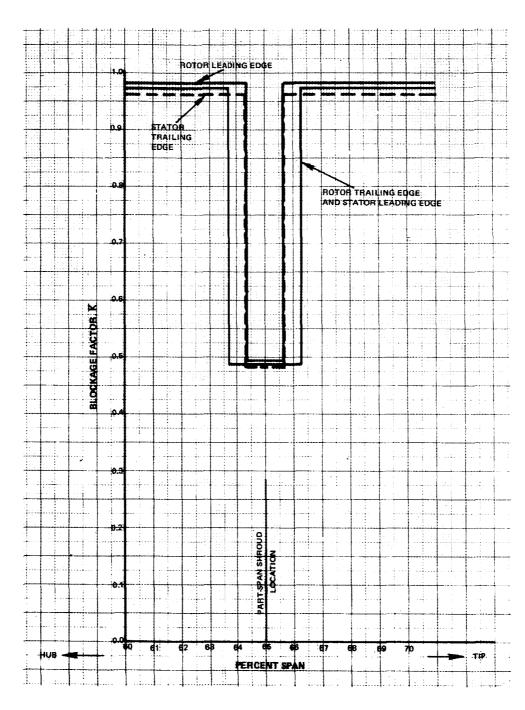


Figure 4 Spanwise Distribution of Local Blockage Factor in Vicinity of Part-Span Shroud

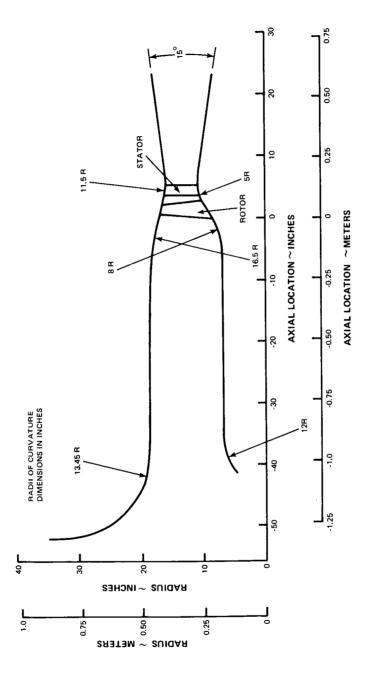


Figure 5 Flowpath

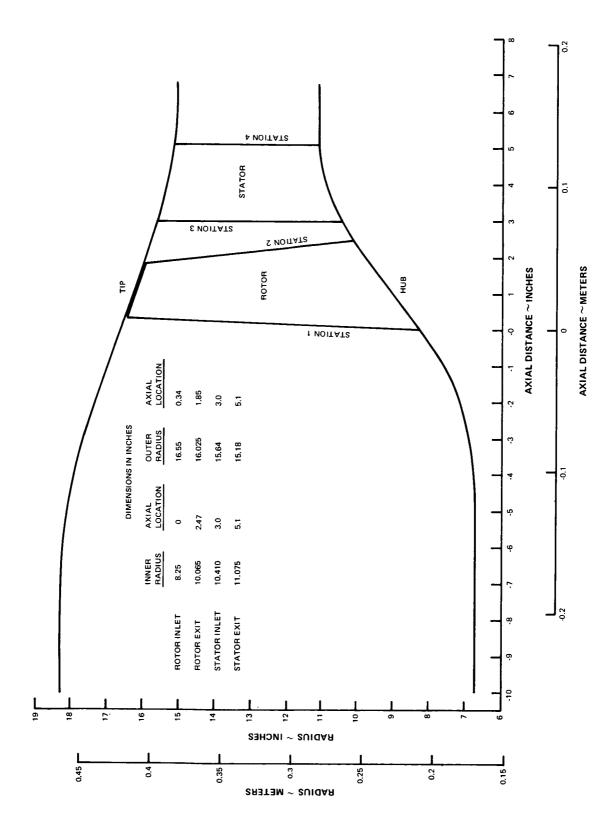


Figure 6 Rotor and Stator Portion of Flowpath

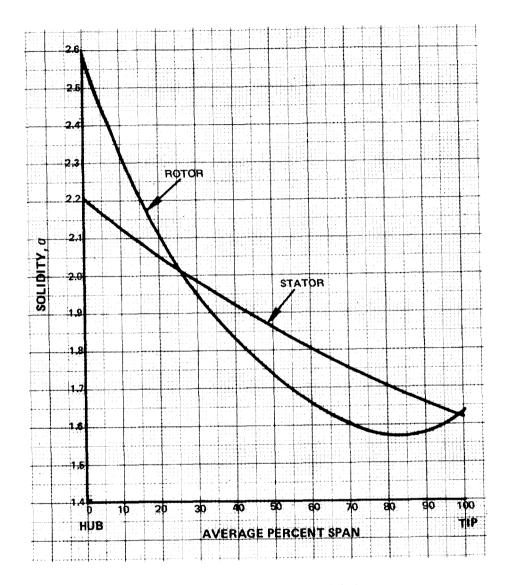


Figure 7 Rotor and Stator Solidities

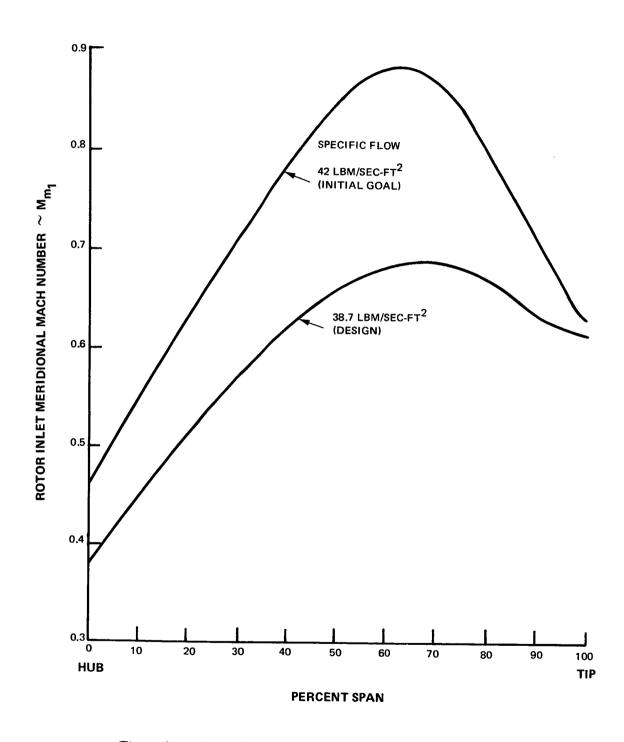


Figure 8 Rotor Inlet Meridional Mach Number Profile

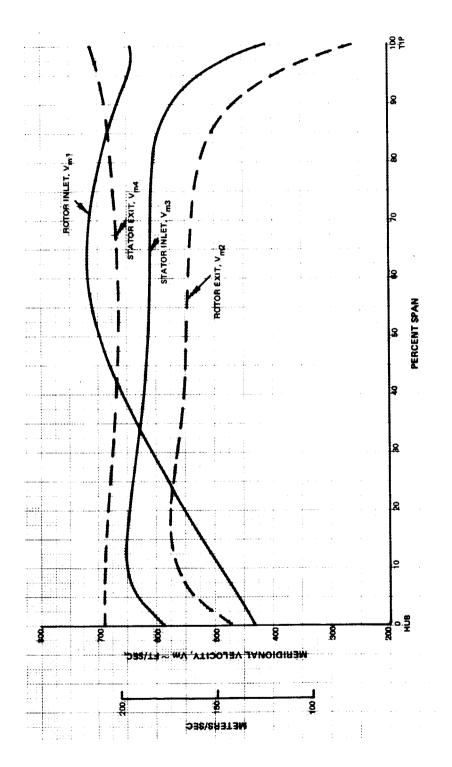


Figure 9 Rotor and Stator Inlet and Exit Meridional Velocity Profiles

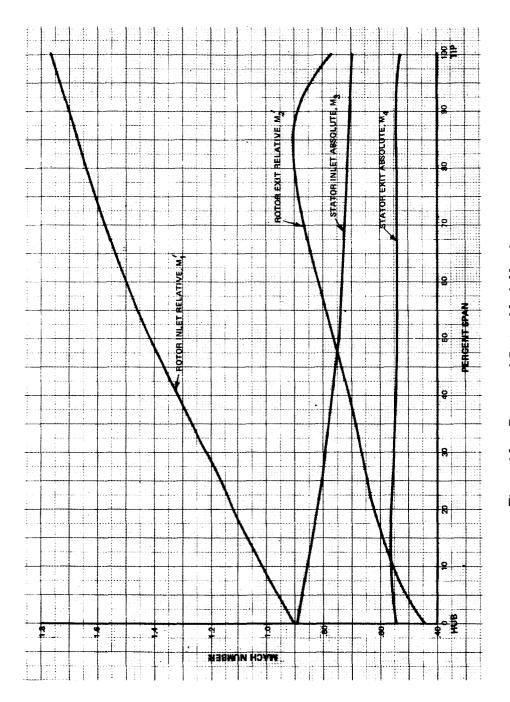
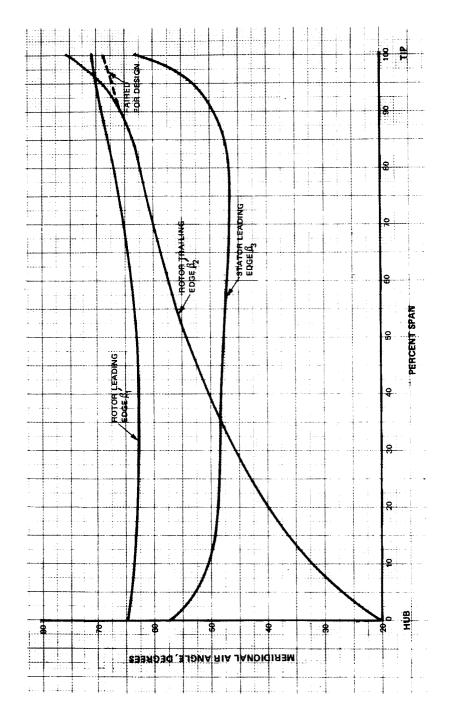


Figure 10 Rotor and Stator Mach Numbers



igure 11 Rotor Inlet and Exit Relative and Stator Inlet Absolute Air Angles

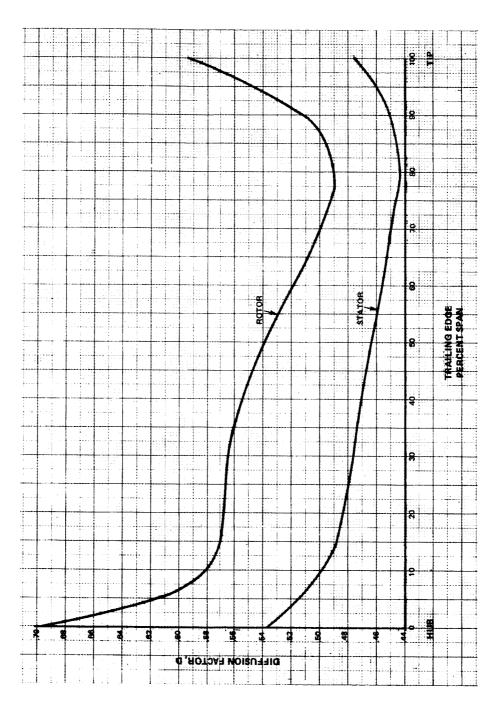


Figure 12 Rotor and Stator Diffusion Factors

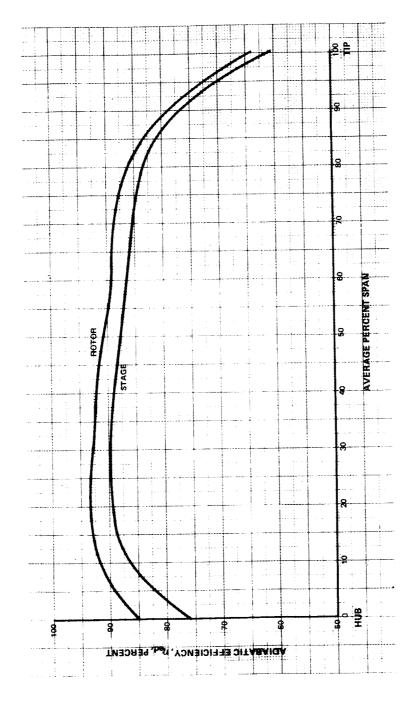


Figure 13 Rotor and Stage Efficiency

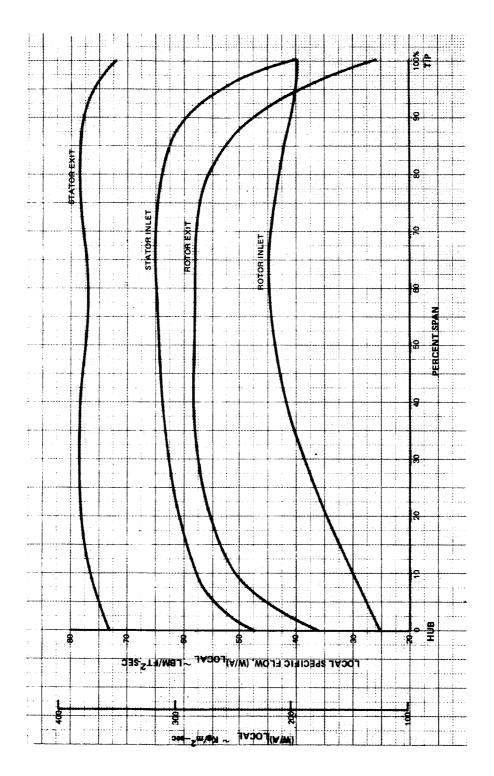
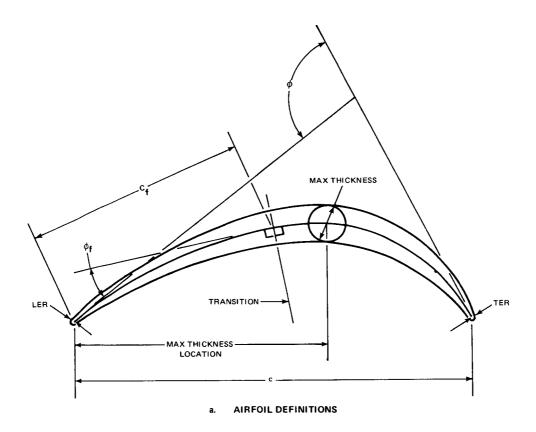


Figure 14 Rotor and Stator Spanwise Specific Flow, W/A = ρ V_Z



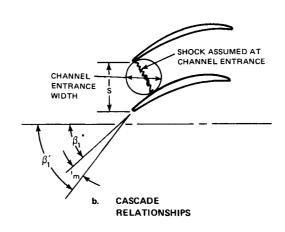


Figure 15 MCA Airfoil Definitions and Cascade Relationships

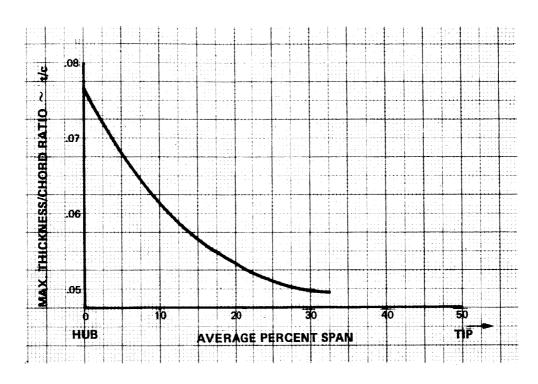


Figure 16 Maximum-Thickness to Chord Ratio for Modified MCA Rotor Blade Sections

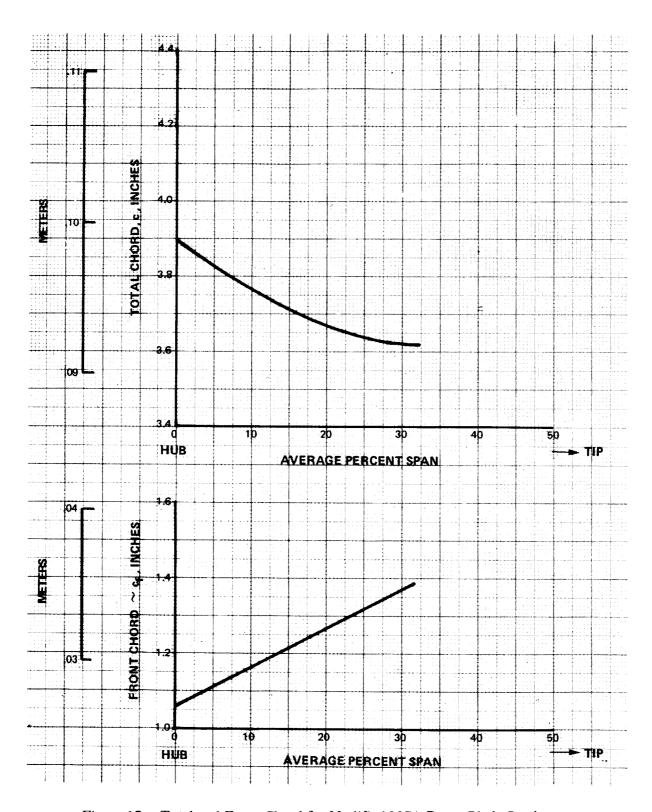


Figure 17 Total and Front Chord for Modified MCA Rotor Blade Sections

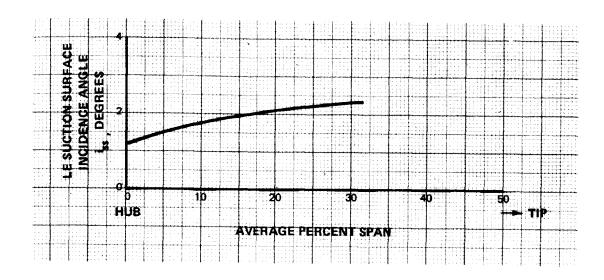


Figure 18 Suction Surface Incidence Angle for Modified MCA Rotor Blade Sections

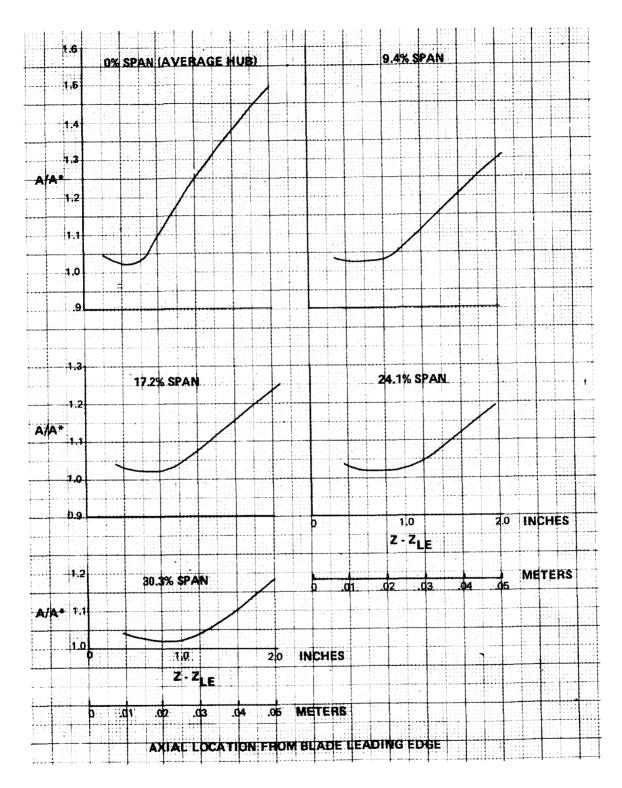


Figure 19 A/A* Distribution for Modified MCA Rotor Blade Sections

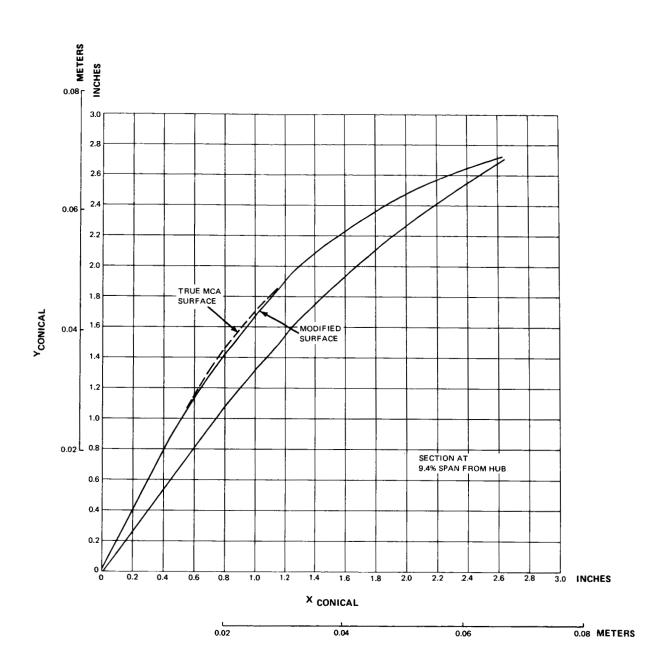


Figure 20 Modified MCA Blade Section on Unwrapped Conical Surface

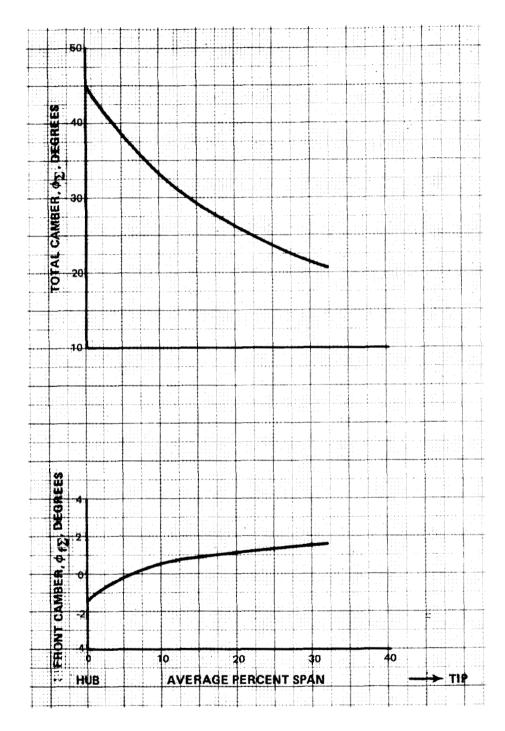


Figure 21 Total and Front Camber Angle Distributions for Modified MCA Rotor Blade Sections

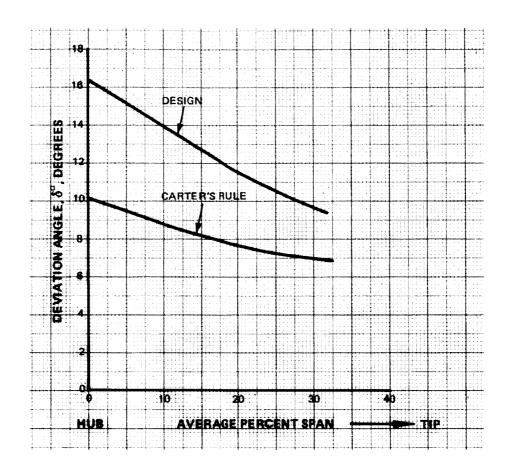


Figure 22 Deviation Angles for Modified MCA Rotor Sections and Comparison with Carter's Rule

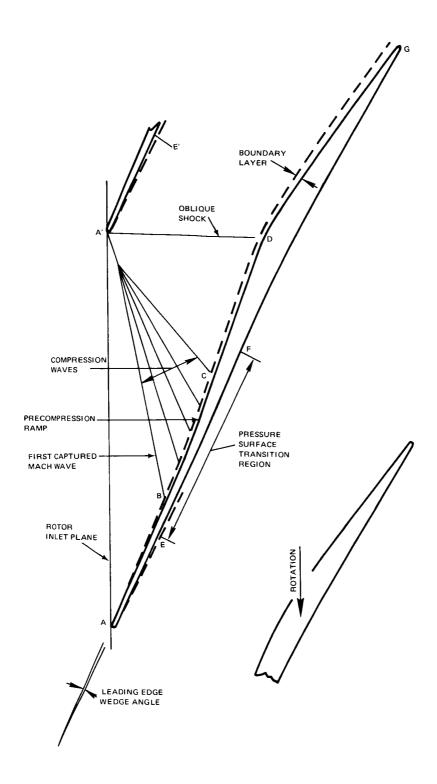


Figure 23 Precompression Blade Airfoil Terminology

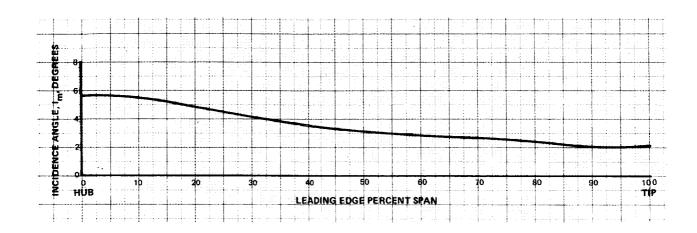


Figure 24 Rotor Blade Mean Camber Line Incidence Angle

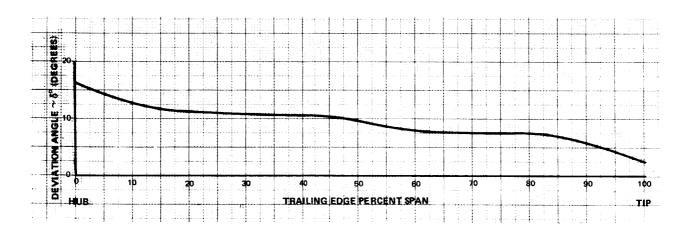


Figure 25 Rotor Blade Mean Camber Line Deviation Angle

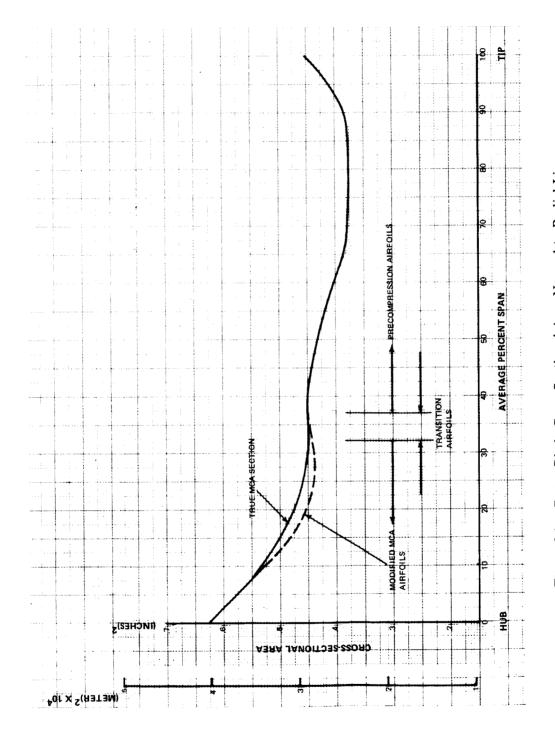
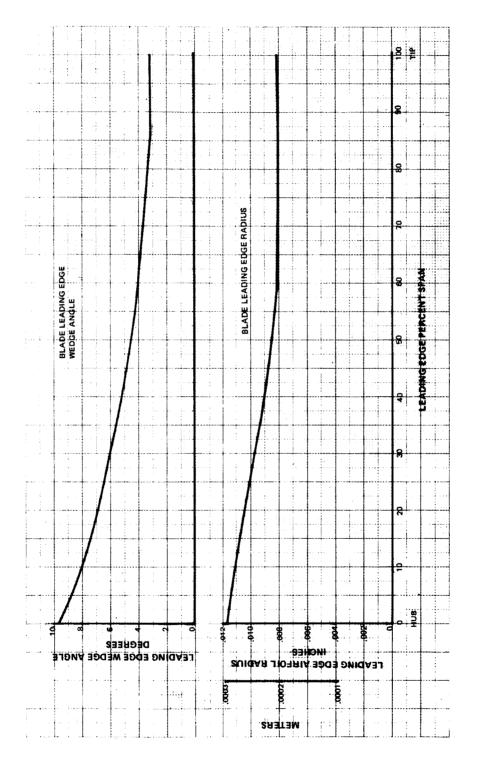


Figure 26 Rotor Blade Cross-Sectional Area Normal to Radial Line



Rotor Blade Leading Edge Wedge Angle and Leading Edge Airfoil Radius Figure 27

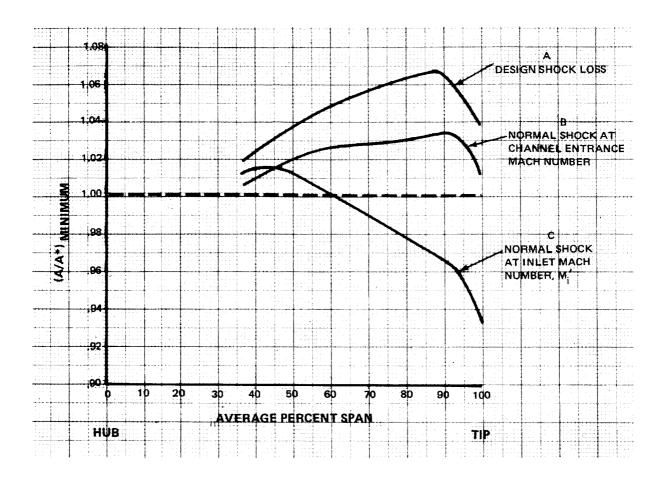


Figure 28 Rotor Blade Spanwise (A/A*)min for Various Assumed Shock Losses

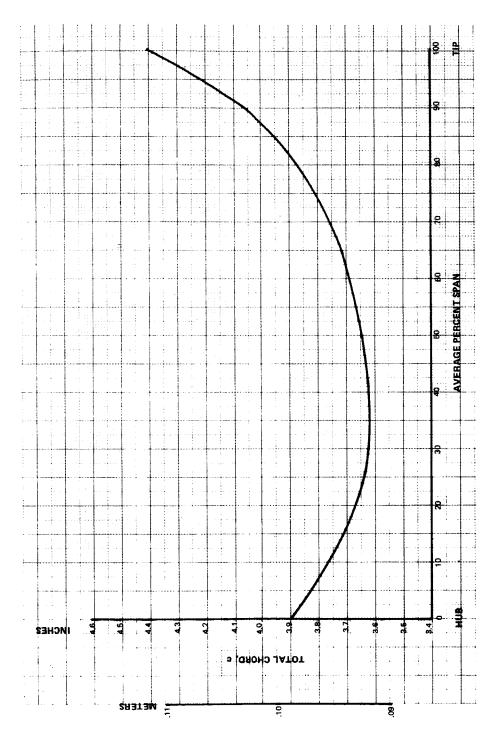


Figure 29 Rotor Blade Chord on Conical Surfaces

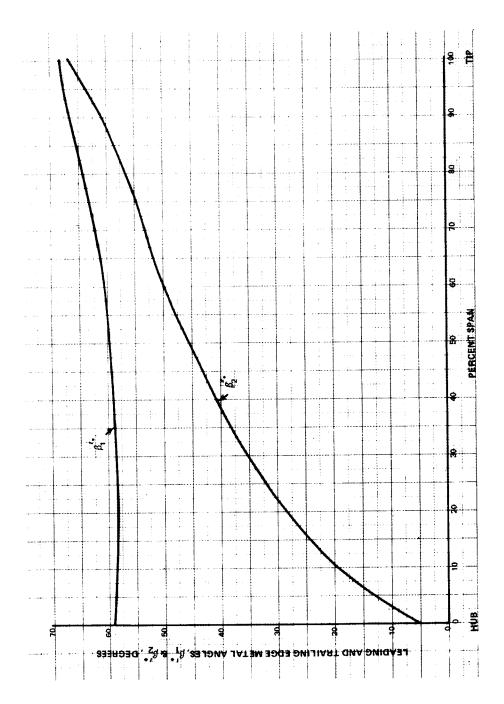


Figure 30 Rotor Blade Leading and Trailing Edge Metal Angles

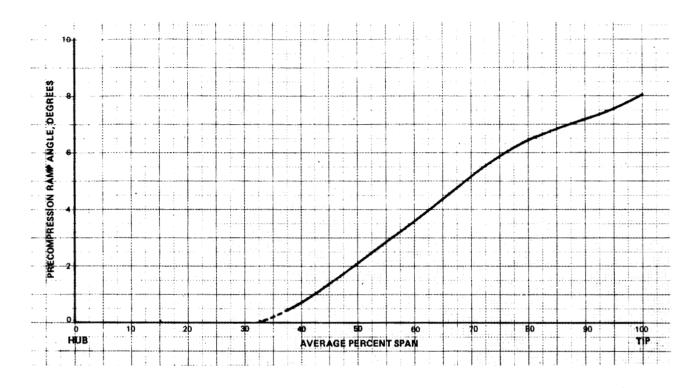


Figure 31 Rotor Blade Precompression Ramp Angle

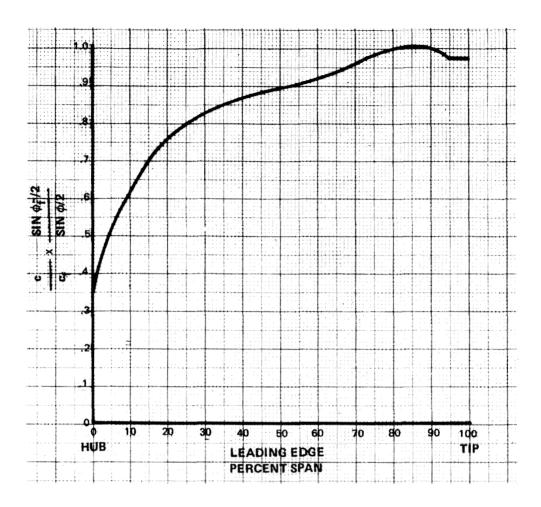


Figure 32 Chord-Camber Parameter for Stator Vane

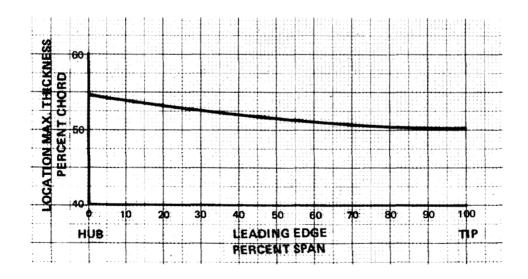


Figure 33 Location of Maximum Thickness for Stator Vane

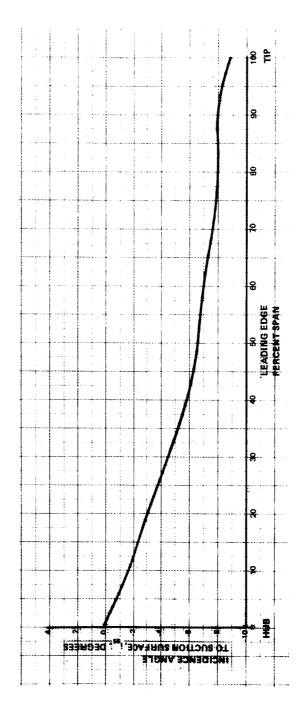


Figure 34 Stator Vane Suction Surface Incidence Angle

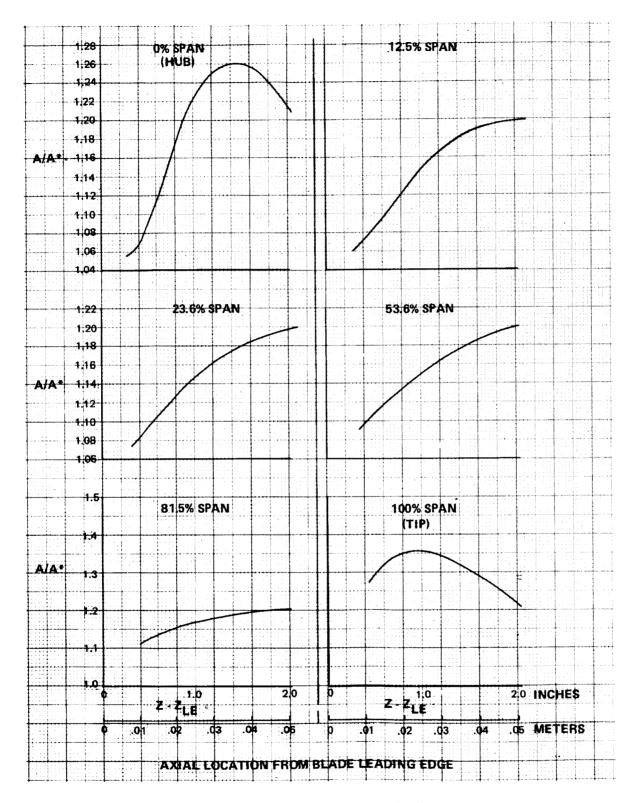


Figure 35 Stator Vane Distribution of A/A*

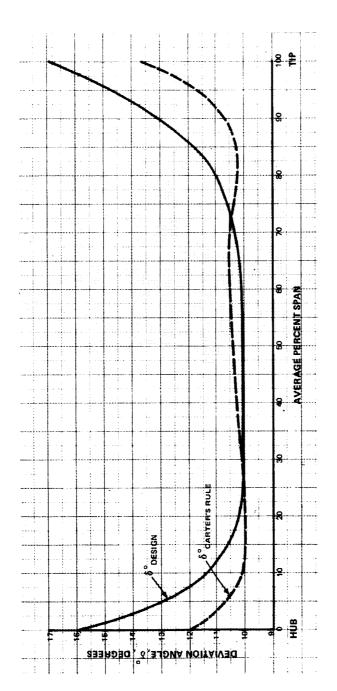


Figure 36 Stator Vane Deviation Angle and Comparison with Carter's Rule

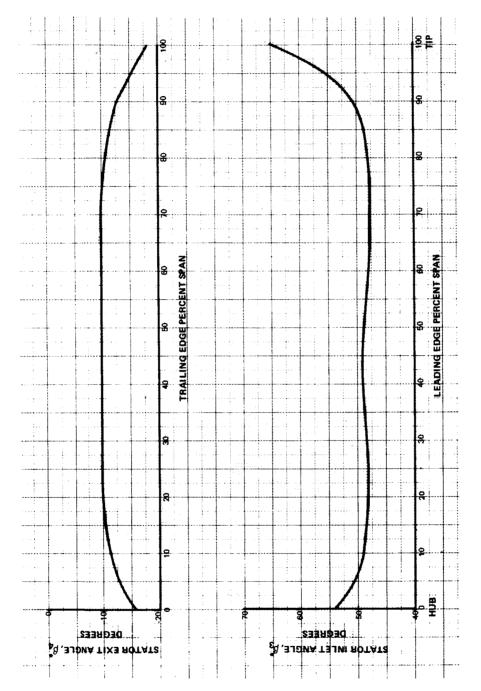


Figure 37 Stator Vane Inlet and Exit Mean Camber Line Metal Angles on Conical Surfaces

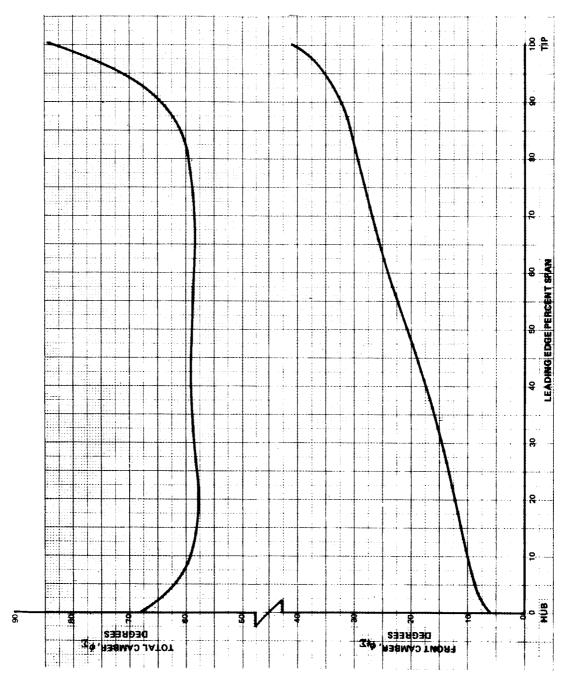


Figure 38 Stator Vane Front and Total Camber Angles

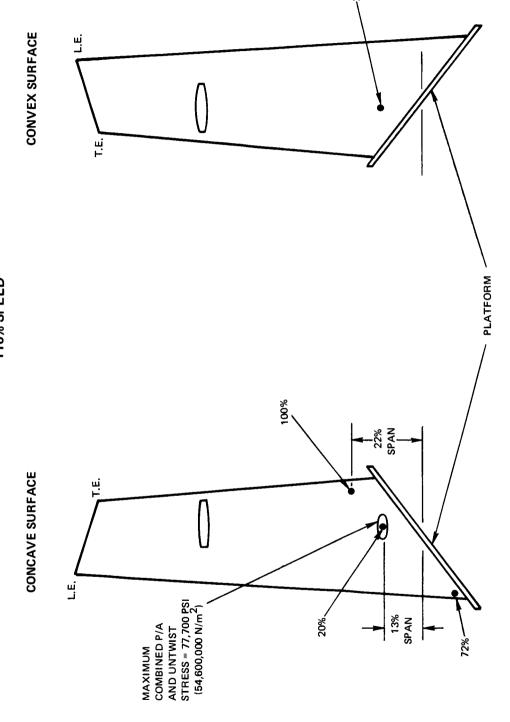


Figure 39 Rotor Blade Maximum Stress Location

AREAS OF HIGH VIBRATORY STRESS (% OF MAX.) IN FIRST BENDING MODE

AREA OF HIGHEST STEADY STRESS

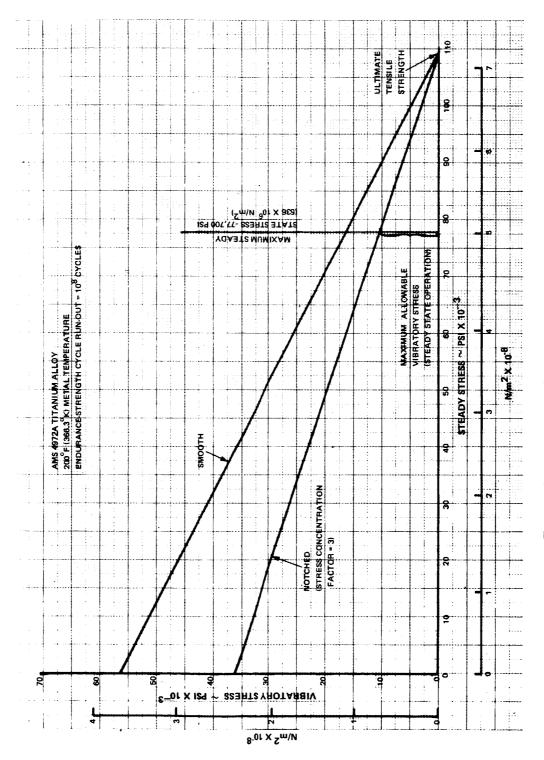


Figure 40 Rotor Modified Goodman Diagram

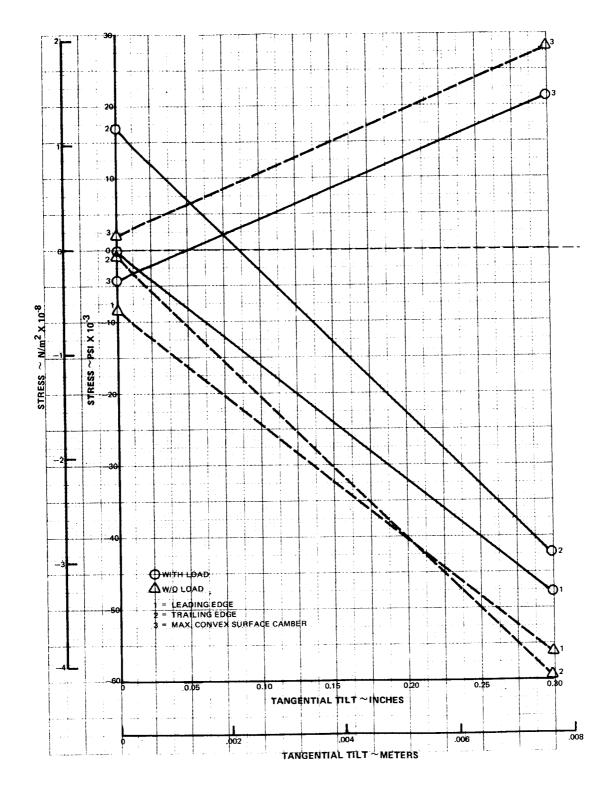


Figure 41 Effect of Tangential Tilt on Rotor Blade Gas Bending Stress

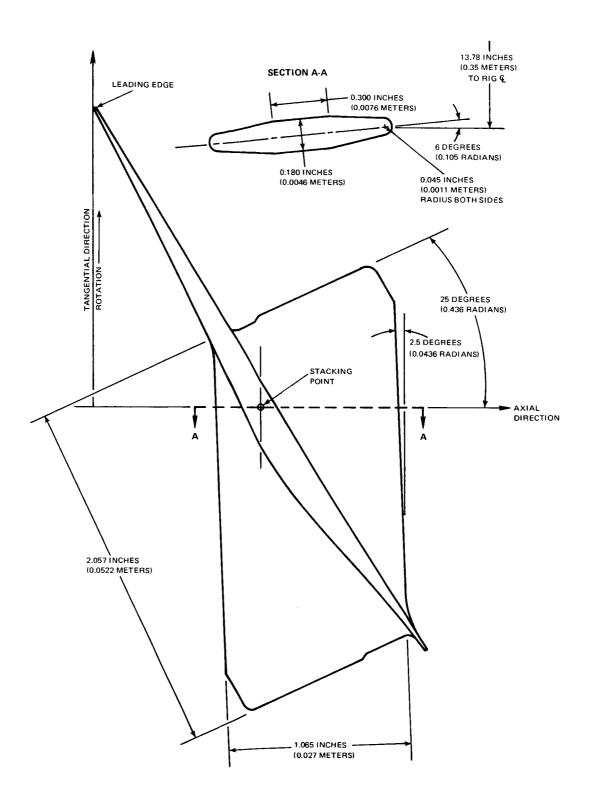


Figure 42 Top View of Rotor Blade Part-Span Shroud

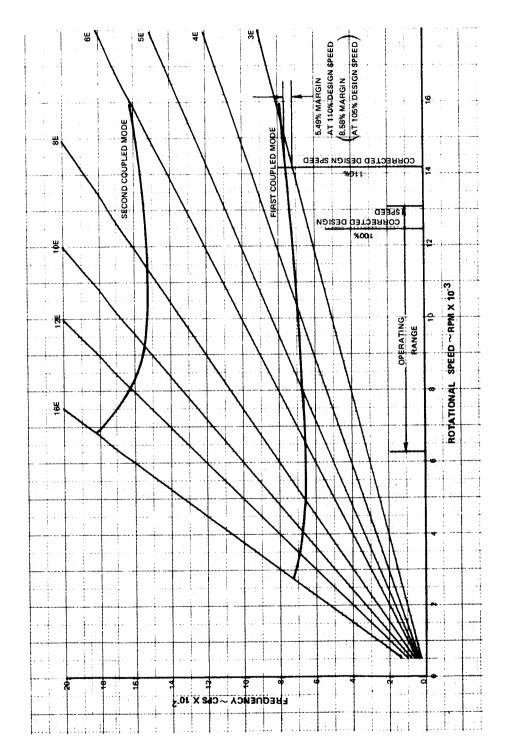


Figure 43 Resonance Diagram for Rotor Blade and Disk

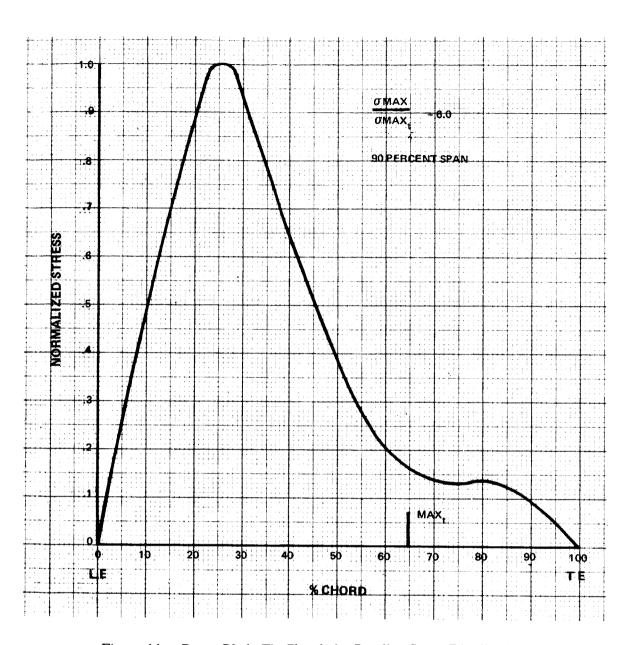


Figure 44 Rotor Blade Tip Chordwise Bending Stress Distribution

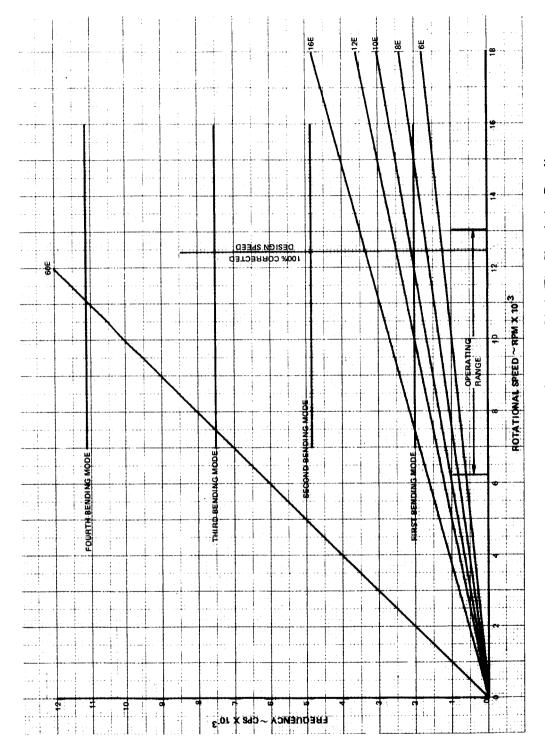


Figure 45 Resonance Diagram for Rotor Blade Tip Chordwise Bending

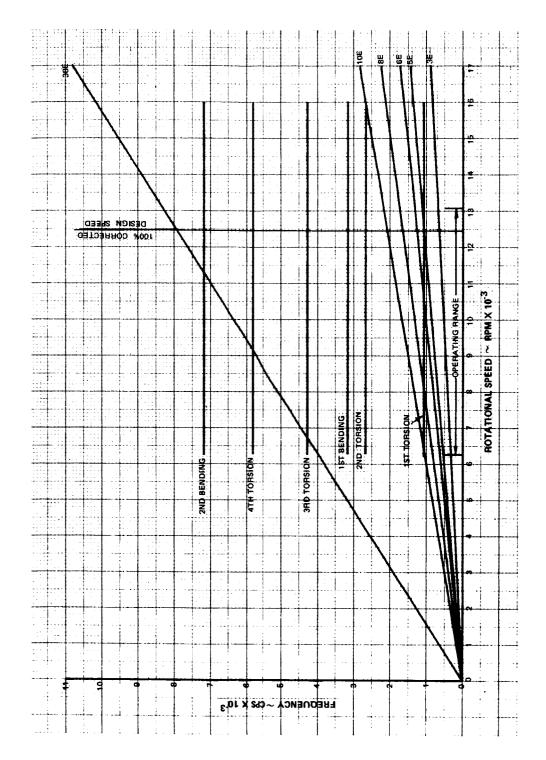


Figure 46 Resonance Diagram for Stator Vane

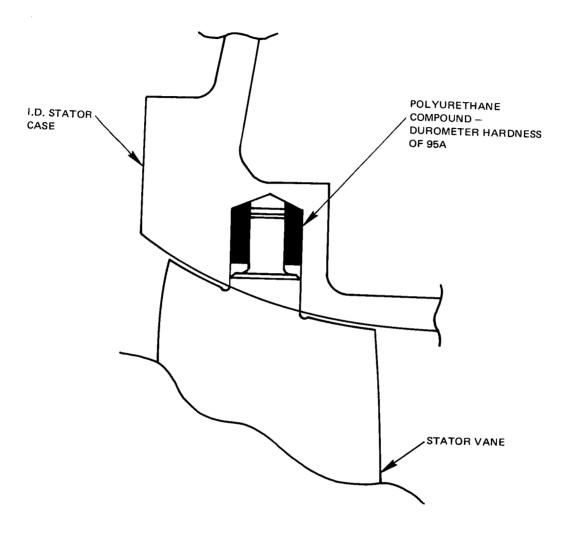


Figure 47 Stator Vane Polyurethane Bushing

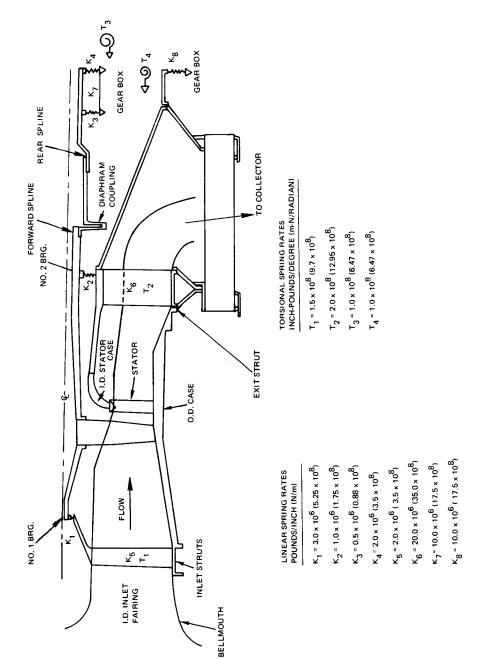


Figure 48 Compressor Rig Spring Location and Spring Rates

A) 1st MODE NATURAL FREQUENCY (RPM) = 5500 INLET INLET STRUTS BELLMOUTH RELATIVE DISPLACEMENT NO. 3 BRG. ROTOR NO. 1 BRG. DIAPHRAGM VANES NO. 2 BRG. EXIT STRUTS COUPLING O.D. CASE I.D. STATOR CASE

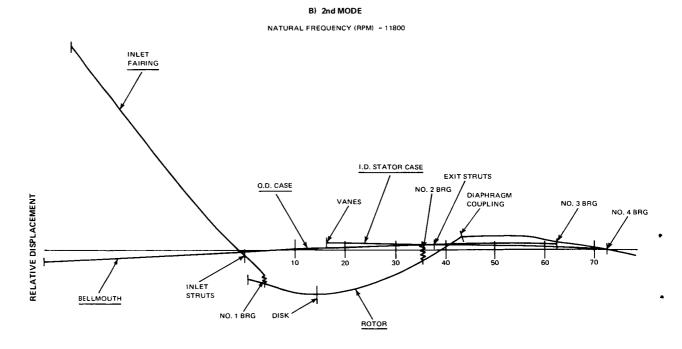
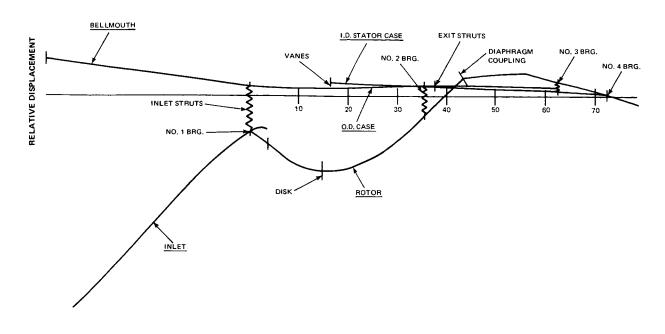


Figure 49 Critical Speed Mode Shapes

C) 3rd MODE

NATURAL FREQUENCY (RPM) = 12500



D) 4th MODE

NATURAL FREQUENCY (RPM) = 14800

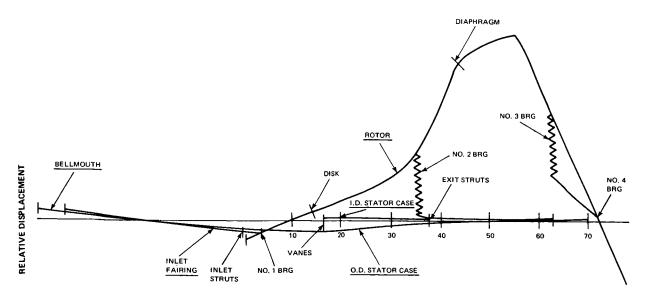


Figure 49 (Cont'd) Critical Speed Mode Shapes

APPENDIX 1

Flow Field Calculation Procedures

The aerodynamic flow field calculation used in this design assumes axisymmetric flow and uses solutions of continuity, energy, and radial equilibrium equations. These equations account for streamline curvature and radial gradients of enthalpy and entropy, but viscous terms are neglected. Calculations were performed on stations oriented at an angle λ with respect to the axial direction.

The equation of motion is in the form of:

$$\frac{1}{2} \frac{\partial V^2_m}{\partial m} \cos(\lambda - \epsilon) + \frac{V^2_m}{R_c} \sin(\lambda - \epsilon) - \frac{V^2_{\theta}}{r} + \frac{1}{\rho} \frac{\partial p}{\partial r} = 0$$

$$R_{\rm C} = \frac{\partial \epsilon}{\partial m}$$
 = streamline radius of curvature

Enthalpy rise across a rotor for a streamline ψ is given by the Euler relationship

$$\Delta H_{Rotor} = (U_2 V_{\theta_2})_{\psi} - (U_1 V_{\theta_1})_{\psi}$$

Weight flow is calculated by the continuity equation

$$W = 2\pi \int_{y \text{ root}}^{y \text{ tip}} \overline{K} \rho V_{\text{m}} \frac{\sin (\lambda - \epsilon)}{\sin \lambda} \quad \text{y dy}$$

where K is the local blockage factor and y is the length along the calculation station from the centerline to the point of interest.

APPENDIX 2 AERODYNAMIC SUMMARY

Tables V and VI

APPENDIX 2

TABLE V
ROTOR AERODYNAMIC SUMMARY

	100(Tip)	1.00 1.00 0.00 0.00 .\$926 1.7696		33.100 639.67 1910.24 639.67	0.00 -1799.95 1799.95 0.00 70.44 -18.68		.8407 194.97 582.24 194.97 0.00 -548.63 548.63 0.00 1.2293																																																																															
	06	1.00 1.00 0.00 0.00 .6149 1.7027		31.440 662.06 1833.43 662.06	0.00 -1709.68 1709.68 0.00 68.83		.7586 201.80 558.83 201.80 0.00 -521.11 521.11 0.00 1.2013																																																																															
	80	1.00 1.00 0.00 0.00 .6461 1.6420	English Units	English Units	English Units	English Units	English Units	English Units	English Units	English Units	English Units	English Units	English Units	English Units	English Units	English Units						29.780 693.19 1761.62 693.19	0.00 -1619.41 1619.41 0.00 66.82 -12.60		.7564 211.28 536.94 211.28 0.00 493.60 493.60 0.00 1.1662 2199																																																													
	70	1.00 1.00 0.00 0.00 .6638 1.5752																	28.120 710.63 1686.33 710.63	0.00 -1529.14 1529.14 0.00 65.06		7143 216.60 514.00 216.60 0.00 466.08 466.08 0.00 1.1355 1379																																																																
	09	1.00 1.00 0.00 0.00 .6651 1.4999																												26.460 711.92 1605.44 711.92	0.00 -1438.87 1438.87 0.00 63.66 -2.90		.6721 216.99 489.34 216.99 0.00 438.57 438.57 0.00 1.1111																																																					
	20	1.00 1.00 0.00 0.00 .6451 1.4134															24.800 693.15 1516.33 693.15	0.00 -1348.60 1348.60 0.00 62.79 2.70		.6299 211.27 462.18 211.27 0.00 411.06 411.06 0.00 1.0959 .0471																																																																		
	40	1.00 1.00 0.00 0.00 .6086 1.3170															English Units	English Units	inglish Units	English Units	English Units	inglish Units	inglish Units	iglish Units	glish Units	nglish Units	nglish Units	23.140 655.71 1418.93 655.71	0.00 -1258.33 1258.33 0.00 62.47 8.80		.5878 199.66 432.49 199.86 0.00 -383.54 383.54 0.00 1.0904 .1536																																																							
Rotor Inlet	30	1.00 1.00 0.00 0.00 .5598 1.2154																										nglish Units	English Units	English Units	nglish Units	nglish Units	nglish Units	inglish Units	English Units	English Units	English Units	English Units	21.480 606.13 1315.97 606.13	0.00 -1168.07 1168.07 0.00 62.59 15.25	SI Units	.5456 184.75 401.11 184.75 0.00 -356.03 356.03 0.00 1.0924 .2662																																												
'	20	1.00 1.00 0.00 0.00 .5076 1.1123																	19.820 552.77 1211.28 552.77	0.00 -1077.80 1077.80 0.00 62.85		.5034 168.48 369.20 168.48 0.00 -328.51 328.51 0.00 1.0969 .3787																																																																
	10	1.00 1.00 0.00 0.00 .4502 1.0082																										18.160 492.78 1103.66 492.78	0.00 -987.53 987.53 0.00 63.49 27.60		.4513 150.20 336.40 150.20 0.00 301.00 0.00 1.1081 .4817																																																							
	0(Hub)	1.00 1.00 0.00 3.3924 .9053																																																																																	16.500 431.62 995.67 431.52	0.00 -897.26 897.26 0.00 64.31 33.99		.4191 131.56 303.48 131.56 0.00 273.48 0.00 1.1224 .5933
	Percent Span																																																																																					
															Inches Ft/Sec Ft/Sec Ft/Sec	Ft/Sec Ft/Sec Ft/Sec Degrees Degrees		Meters m/Sec m/Sec m/Sec m/Sec m/Sec m/Sec m/Sec Redians Radians Radians																																																																				
		Total Pressure Ratio* Total Temperature Ratio* © D M M'		Diameter V V V'	ψ		Diameter \mathbf{v}'																																																																															

*Calculations based on an inlet pressure of 2,116 psf and an inlet temperature of 518.6°R

APPENDIX 2

TABLE V (Cont'd)

ROTOR AERODYNAMIC SUMMARY

	100(Tip)	2.34 1.4282 .3643 .594 .6305		32.050 808.84 1007.66 254.33 767.82 -975.04 1742.85 71.67	-19.3	.8141 246.54 307.14 307.14 77.52 234.03 -297.19 531.22 1.2509 1.3156 3370																																																
	06	2.34 1.3576 2304 511 .6495			30.858 810.81 1113.59 463.29 665.40 -1012.64 1678.03 55.15	-16.2	7838 247.13 339.42 141.21 202.81 -308.65 511.47 9626 1.1417																																															
	80	2.34 1.3266 .1595 .490 .6640		29.666 817.98 1110.40 519.22 631.88 -981.33 1613.21 50.59	-12.0	.7535 249.32 338.45 158.26 192.60 -299.11 491.71 .8830 1.0840																																																
	70	2.34 1.3146 .1324 .500 .6783	English Units								28.474 830.43 1059.97 538.21 634.11 -914.29 1548.39 49.78 59.60	-7.40	.7232 253.12 323.08 163.44 193.28 -278.68 471.95 .8688 1.0403																																									
	09	2.34 1.3072 .1171 .519 .6927 .8189															27.282 844.21 998.08 543.18 645.26 -837.32 1483.57 49.95	-3.0	.6930 257.32 304.21 165.56 196.98 -255.22 452.19 .8718 .9953																																			
	20	2.34 1.3036 .1141 .538 .7096												26.090 861.84 927.82 544.84 667.77 -750.99 1418.75 50.79 54.04	2.1	.6627 262.69 282.80 1.166.07 203.54 -228.90 432.44 .8864 .9431																																						
	40	2.34 1.3001 .1109 .554 .7313		24.898 884.53 861.62 551.15 691.83 -662.10 1353.93 51.46	7.20	.6324 269.61 262.82 167.99 210.87 -201.81 412.68 .8981 .8763																																																
Rotor Exit	30	2.34 1.2967 .1087 .564 .7571		English Units	English Units	English Units	English Units	English Units	English Units	English Units	English Units	English Units	English Units	English Units	English Units	English Units	English Units	lish Units	dish Units	glish Units	glish Units	glish Units	lish Units	glish Units	glish Units	glish Units	glish Units	23.706 911.38 800.41 561.02 718.23 -570.88 1289.11 52.01 45.50	12.80 SI Units	.6021 277.79 243.96 171.00 218.92 -174.00 392.92 .9077 .7941																								
×	•	2.34 1.2956 1.165 5.67 7884																22.514 944.45 739.28 567.98 754.48 469.81 1224.29 53.01	19.10	.5719 225.33 225.33 173.12 229.97 -143.20 333.17 9252 6868 .3334																																		
	o	2.34 1.2984 1.530 5.81 .8217													21.322 980.92 666.36 563.03 803.19 -356.28 1159.47 54.97	26.30	.5416 299.98 203.11 171.51 244.81 -108.59 353.41 9594 -5639 -4590																																					
	0(Hub)	2.34 1.3228 .3550 .695 .8623																																																		20.130 11032.82 499.82 469.48 920.46 -174.20 11094.65 63.03 20.40	34.89	.5113 314.80 152.35 142.79 280.56 -53.10 333.65 1.100 .5090
	Percent Span																																																					
				Inches Ft/Sec Ft/Sec Ft/Sec Ft/Sec Ft/Sec Degrees	Degrees	Meter m/Sec m/Sec m/Sec m/Sec m/Sec m/Sec m/Sec Radians Radians Radians																																																
		Total Pressure Ratio* Total Temperature Ratio* \overline{\infty} D M' M'		Diameter V V V V V V V V V V V V V V V V V V V	L W	Diameter V V V V V V V V V V V V V V V V V V V																																																

 $\mbox{*}$ Calculations based on an inlet pressure of 2,116 psf and an inlet temperature of 518.6°R

APPENDIX 2

TABLE VI

STATOR AERODYNAMIC SUMMARY

Stator Inlet

	<u>a</u> 97	Percent Span 0(1		20	30	40	20	09	70	80	06	100(Tip)
Total Pressure Ratio* Total Temperature Ratio* \$\overline{D}\$ \$M\$		22.7.1.1.2.2.8.8.8.8.8.8.8.8.8.8.8.8.8.8.8.8	2.34 1.3228 0.00 0.00 .8924 .5296	2.34 1.2983 0.00 0.00 .8557 .6439	2.34 1.2956 0.00 0.00 .8229 .6869	2.34 1.2966 0.00 0.00 .7924	2.34 1.2999 0.00 0.00 7.681	2.34 1.3032 0.00 0.00 .7481	2.34 1.3067 0.00 0.00 .7337 .8444	2.34 1.3135 0.00 0.00 .7237 .8852	2.34 1.3234 0.00 0.00 .7159	2.34 1.3524 0.00 0.00 .7092	2.34 1.4282 0.00 0.00 .6940 .7845
					щ	English Units							
Diameter V V V V,	Inches Ft/Sec	825	20.820 1063.98	21.866 1016.45	22.912 981.01	23.953 949.25	25.004	25.050	27.096	28.142	29.188	30.234	31.280
• * • • • • • • • • • • • • • • • • • • •	Ft/Sec Ft/Sec		3.13	764.79 648.06 783.04	642.72	864.31 629.32	615.82	608.84	607.32	605.39	601.28	566.99	998.80 402.15
	Ft/Sec Ft/Sec	27.5	3.35 42.23 37.13	783.04 406.02 1189.06	/41.11 -504.82 1245 94	/10.66 -592.16 1302.82	-671.30	-748.49	-823.74 -823.74 1473.46	639.52 -890.82 1530.34	635.89 -951.33 1587.77	669.16 -974.94 1644.10	/86./2 -914.26
æ æ	Degrees Degrees	. 56 20	55	50.39	49.06	48.47	48.14	47.66	46.93	46.57	46.00	49.72	62.93
L W	Degrees	30.	.42	22.20	16.10	10.80	6.50	2.90	0.10	-3.7	-7.0	.10.1	-17.05
						SI Units							
Diameter V	Meters	.53	5288	.5554	.5820	.6085	.6351	.6617	.6882	.7148	7414	.7680	.7945
> >	m/Sec	92	4.30 2.46	309.82 233.11	249.01 249.65	289.33 263.44	281.73 277.96	2/5.51 294.13	311.98	328.32	266.76 343.04	343.78	269.30 304 44
v _m	m/Sec	17	7.74	197.53	195.90	191.32	188.01	185.57	185.11	183.27	172.82	172.82	122.57
$\dot{\theta}_{\Delta}$	m/Sec	27	1.26	238.67	225.85	216.61	209.82	203.63	198.03	194.93	193.82	203.96	239.79
ė.	m/Sec	-73	83	123.75	-153.87	-180.49	-204.61	-228.14	-251.08	-271.52	-289.97	-297.10	-278.67
) «	m/Sec	4 9	5.09	362.43	379.76	397.10	414.44	431.77	449.11	466.45	483.79	501.12	518.46
50° 500	Kadians Radians	₹. &	70/	8794 5596	.8582	.8460	.8402	8317	.8191	.8128	.8134	.8678	1.0983
ኒ Ψ	Radians	53	608	3875	.2810	.1885	.1135	.0506	.0018	.9/36 0646	1222	1763	2976

* Calculations based on all inlet pressure of 2,116 psf and an inlet temperature of 518.6° R

APPENDIX 2

TABLE VI (Cont'd)

STATOR AERODYNAMIC SUMMARY

Stator Exit

00(Tip)	2.2860 1.4282 .0640 .476 5486 1.3886		30.360 709.96 709.96 709.96 5.00 1660.95 1660.95 5.00 66.73	7712 216.40 215.40 215.40 503.21 503.21 503.21 503.21 1.1647
			29.539 33 883.15 77 1745.54 17 583.15 77 5000 0 1606.31 11 1606.31 11 1606.31 17 6000 0 5000 6 65.36 6	7503 7503 7503 7503 7508.22 208.22 208.22 20.00 6489.60 5600 5
8	2.2969 1.3487 .0646 .451 .5427 1.3865		29 683 683 683 683 683 683 683 683 683 683	757. 208 208 208 208 209 489 489 6.00 1.1-1
80	2.3035 1.3215 .0540 .444 .5399 1.3641		28.718 673.08 1700.54 673.08 0.00 -1561.66 0.00 66.68	.7294 205.16 518.33 205.16 0.00 476.00 476.00 1.1638 0628
70	2.3032 1.3132 .0535 .450 .5369 1.3332		27.897 667.47 1657.38 667.47 0.00 -1517.02 1517.02 0.00 66.25	.7086 503.46 505.17 203.45 0.00 462.39 60.00 1.1563
09	2.3002 1.3066 .0565 .456 .5353 1.3023		27.076 663.92 1615.15 663.92 0.00 -1472.37 1472.37 0.00 65.73	.6877 202.36 492.30 202.36 0.00 -448.78 448.78 0.00 1.1471
ţ	2.2956 1.3033 .0612 .464 .5356 1.2710		26.255 663.38 1574.33 663.38 0.00 -1427.73 1472.73 0.00 65.08	.6669 202.20 202.20 202.20 0.00 435.17 435.17 0.00 1.1358 .0140
40	2.2900 1.3001 .0662 .471 .5384 1.2412		25.434 665.85 1535.03 665.85 0.00 -1383.08 1383.08 64.29 2.10	.6460 202.95 467.88 202.95 0.00 421.56 0.00 1.1.221 0.367
30	2.2792 1.2968 .0787 .476 .5419 1.2118	inglish Units	24.613 669.17 1496.40 669.17 0.00 -1338.44 0.00 63.44 3.10	.6252 203.96 456.10 203.95 0.00 407.96 407.96 0.00 1.1072
20	2.2669 1.2957 .0871 .484 .5514 1.1854	ш	23.792 679.97 1461.77 679.97 0.00 -1293.79 1293.79 62.26	.6043 207.26 445.55 207.26 0.00 -394.35 94.35 0.00 1.0866
10	2.2427 1.2977 .1094 .499 .5571 1.1559		22.971 687.04 1425.62 687.04 0.00 -1249.14 1249.14 0.00 61.19	.5835 209.41 434.53 209.41 0.00 -380.74 380.74 0.00 1.0679 .0873
O(Hub)	2.1861 1.3228 1.629 538 5528 1.1138		22.150 688.54 1387.41 686.54 0.00 -1204.50 1204.50 0.00 60.25	.5625 209.87 422.88 209.87 0.00 -367.13 367.13 0.00 1.0515
Percent Span				
			Inches Ft/Sec Ft/Sec Ft/Sec Ft/Sec Ft/Sec Ft/Sec Ft/Sec Ft/Sec Degrees Degrees Degrees	Meters m/Sec m/Sec m/Sec m/Sec m/Sec m/Sec M/Sec M/Sec Sec M/Sec M
	Total Pressure Ratio* Total Temperature Ratio* © D M M'		Diameter V V' V' V' V' V' V' V'	Diameter \mathbf{v}'

 * Calculations based on an inlet pressure of 2,116 psf and an inlet temperature of 518.6°R

APPENDIX 3

COSINE VARIATION OF BLADE CHANNEL AREA

The suction surface D-G of Figure 23 is obtained by knowing the pressure surface shape and the local channel areas determined by the equation

$$A = A_D + (A_G - A_D) \left[1 - \cos \left(\frac{\pi}{2} \frac{Z - Z_D}{Z_G - Z_D} \right) \right]$$

where $A_D = A_D(Z)$

This function is calculated assuming constant corrected specific flow from core-flow conditions downstream of the oblique shock at D

and $A_G = A_G(Z)$

This function is calculated assuming constant corrected specific flow from exit core-flow conditions

APPENDIX 4

BLADE AIRFOIL GEOMETRY ON CONICAL SURFACES

TABLES VII AND VIII

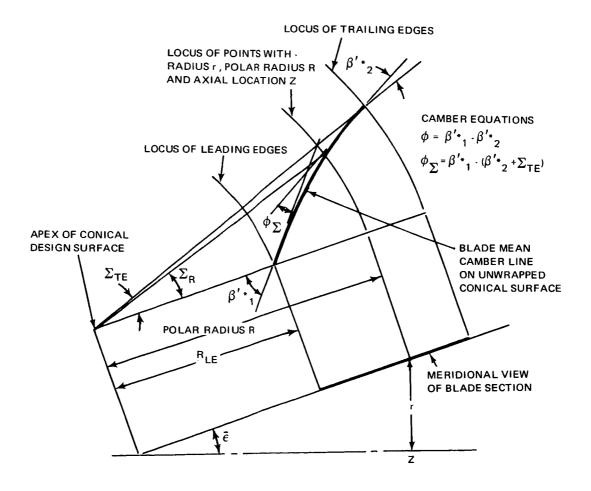


Figure 50 Unwrapped Conical Surface Definitions

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4	

TABLE VII

ROTOR BLADE GEOMETRY ON CONICAL SURFACES

SOLDING OF

intel Hub Diameter 16.5 Inches (.419m) Intel Tip Diameter 33.0 Inches (.838m) Exit Hub Diameter 20.13 Inches (.511m) Exit Tip Diameter 32.11 Inches (.816m)

	0	2.576	2.194	2.067	896.	1.802	0.7	027.1	100.1	0001	0.70.1	1.603	900	0.00	0/5:1	000	.00	1.635																					
Location	t/c Max.	53.7	55.0	55.5	56.0	1.60	1 2 2	5.5	62.6	64.4	† 0 † 4	0.50	0.50	6,50	6.00	6.6.3	66.3	65.5																					
ž	t/c Max.	0770	.0563	.0520	.0498	0561	0546	05.35	5050	3130	5150°	9050	0.00	2440	0460	040	6/40	0475																					
Precompression Blade Pressure Surface Points E	IJ				71.7	970	.000. 585	564	540) C S	000		503	100	37.5	2,45	900	365																					
Precom Pressur E	إد ا				מננ	173	5 191	5	3 3	1091	170	231	001	300	02.0	363	i e	5.0.5																					
Ø	Degrees	8.98	6.47	5.41	6.55	1.6	202	~ =	77) <u>-</u>	195	0.7	91.5	7 7	200	¥2.0-	4.84		×	Radians	.1567	.1344	.1130	.0944	.0760	0326	00.0	.0023	0129	.0250	0377	0456	0529	0724	0827	.0915	0843	
. K u	Degrees	36.30	26.23	25.05	57.87	0.07	717	19	02.5	97.9	45.0	1911	13.48	01.01	.20.42	ر) در.	1 0	00.61-		M	Radians	6334	5418	.4577	3848	3185	1412	.0883	-0106	. 0576	-1106	-1670	2026	-,2352	3174	3563	-,3842	-,34.2U	
Precompression Ramp Angle	Degrees					89.7	3.15	19.0	3.61	60.4	5.5	57.6	2.5	100	, , ,		£C.7	16:1		Precompression Ramp Angle	Radians						.0293	15.40.	0550	0764	0860	1560	1031	.1106	.1211.	.1260	.1316	1881.	
É	Degrees	-1.34	1.00	1.20	1.50															í	Radians	-0724	200	.01745	.0209	.0262													
-{	PE Degrees	44.92	28.82	25.10	22.00	18.30	16.80	26.45	90.4	96:	171	69.71	R 2	13.38	13.50	9:10	(10.1)	6.15		ŧ	Radians	26.30	0009	50.39	4380	.3839	.3193	.2932	609	6000	2027	216	1566	1335	1356	2174	1771.	.1073	
3	W.A. Degrees	7.6	7.1	6.5	5.9	4.6	₹:	- :	3.9	3.8	3.6	4.	r 1 :	3.0	3.0	3.0	3.0	3.0		3	Radians	1	1693	1339	<u> </u>	.1030	.0803	8920	.0715	890.	.0663	0.00.0	0558	0524	.0524	.0524	.0524	.0524	
*8	β ₂ Degrees	4.85	23.90	27.66	31.90	38.98	41.47	44.4	46.43	48.74	50.87	51.85	52.94	53.78	56.53	59.07	62.59	66.43		*7	Radians	, ,	0840	1000	4827	5567	.6803	7237	2277.	8103	.8506	88.8	6 6	9386	9866	1.0309	1.0923	1.1592	
**	Degrees	58.75	58.19	58.17	58.25	59.15	59.43	59.87	96.09	06:09	61.55	62.38	63.23	5 4.13	65.88	66.79	67.50	67.73		*7	β l Radians		1.0.5	1.0163	15101	1.0165	1.0322	1.0371	1.0447	1.0533	1.0627	0.0/0.1	1.0885	501	1.1131	1.1655	1.1779	1.1819	
i i	TER	.0102	6000 9600	.0094	1600	.0124	.0080	.0083	6,000	.0083	.0097	9200.	.0111	1800.	.0064	.00%0	9200.	.0067			TER		000259	000251	000024	000231	000315	.000203	.000211	000201	.000211	.000246	.000193	797000	0007000	000003	.000193	0.1000.8	
	LER	.0117	9010	1010	7600	9800	.0084	.0082	1800	0080	00800	0800	0800	0800	0800	0800	0800	0800			LER Meters		.000297	.000282	73000	00023	81,000	000213	.000208	.000206	.000203	.000203	.000203	00000	0000	00000	0000	.000203	
	Cf Inches	1.055	1.140	1.302	1.344																c _f Meters		.02680	.02896	90800	03307	t 1.co.												
	c Inches	3.900	3.800	3,660	3.620	3.640	3.660	3,665	3.677	3.690	3.720	3.750	3.790	3.842	3.983	4.084	4 220	4.400			c Meters	5 101011	90660	.09652	.09449	36760	26160	96260	06306	.09340	.09373	.09449	.09525	.09627	.09759	101.	61201	.11278	
Diameter	TE		20.966																	Diameter	TE	Meters	.5113	5325	.5508	.5679	2842	102.0	.6541	.6705	.6845	.6982	.7118	.7261	7405	7675	7967	.8156	
Diameter	LE	16.500	18.042	20.480	21.499	24.152	24.944	25.704	26.437	27.147	27.861	28.571	29.231	29.879	31.148	31.772	32 301	33.000		Diameter	LE Metess	Merers	.4191	.4583	.4913	.5202	.5461	6336	6529	6715	5689	707.	7257.	.7425	.7589	.7912	0/08. 77.09	.8382	

APPENDIX 4

TABLES VIII

STATOR VANE GEOMETRY ON CONICAL SURFACES

60 Vanes

Inlet Hub Diameter 20.82 Inches (.529m) Inlet Tip Diameter 31.28 Inches (.7945m) Exit Hub Diameter 22.172 Inches (.563m) Exit Tip Diameter 30.252 Inches (.768m)

Q	2.204 2.151 2.103 2.061 2.061 2.061 1.834 1.736 1.694 1.655 1.635	
a/c	532 521 521 517 517 509 509 508 508 549 449 449 488	
Location of t/c Max. (% c) a/c	\$4.4 \$4.6 \$3.6 \$2.9 \$2.9 \$1.3 \$0.9 \$0.0 \$0.0	
t/c Max.	.0500 .0513 .0513 .0525 .0536 .0567 .0567 .0660 .0660 .0679 .0688 .0700	
Σ Degrees	1.72 1.29 1.06 8.8 8.7 5.1 5.1 5.1 6.21 6.33 6.46 6.56 6.79	E Radians .030 .030 .023 .019 .016 .019 .005 .005 .0009 .0009 .0006 .0009
F Degrees	17.57 15.19 13.26 11.57 10.00 7.06 4.36 1.77 82 368 6.12 8.66 -10.00	Radians 3066 2651 23314 20119 1745 20119 2
φ _{ΓΣ} Degrees	6.0 9.2 10.5 11.9 13.1 15.6 18.6 224.8 224.8 22.6 33.1 41.0	φτΣ Radians
	68.28 61.01 58.44 57.61 57.61 57.65 58.70 58.50 58.31 59.58 64.66 64.66 71.76	φΣ Radians 1.1915 1.0646 1.0198 1.0063 1.01089 1.01243 1.0175 1.0175 1.1283 1.2522 1.4621
W.A. Degrees	5.80 6.21 6.80 7.35 8.03 9.21 10.40 11.34 12.53 12.53 13.12 13.06	M.A. Radians 1.1012 1.1084 1.1187 1.187 1.1401 1.1607 1.1815 1.1919 2.2096 2.214 2.228
β* Degrees	-16.00 -12.50 -11.00 -10.00 -10.00 -10.00 -10.12 -10.30 -11.23 -13.40 -16.00	β 4 Radians -2792 -2182 -2182 -1920 -1798 -1745 -1745 -1766 -1850 -1850 -1939 -2732
β_3^* Degrees	0.48	β [*] Radians 9423 8690 8463 8446 8446 8551 8551 84551 8465 8289 8289 8289 9632
TER	.00500 .00520 .00533 .00533 .00570 .00600 .00660 .00687 .00715 .00711 .00768	TER Meters
LER	.00500 .00520 .00533 .00553 .00570 .00660 .00660 .00687 .00711 .00781	LER Meters .000137 .000137 .000144 .000145 .000168 .000168 .000198 .000198
cf Inches	727 709 708 719 736 810 920 1.180 1.280 1.300 1.300	Gf Meters 01847 01801 01798 01826 01826 01826 01826 01826 01829 02337 02997 03302 03302
c Inches	2.480 2.490 2.490 2.502 2.502 2.532 2.545 2.545 2.556 2.560 2.580 2.590 2.590 2.590	c Meters
Diameter TE Inches	22.17 23.20 23.20 23.66 24.10 24.96 25.38 25.38 27.34 28.83 29.83 30.25	Diameter TE Meters Aeters 5631 5871 5893 6609 6121 6340 6348 6751 6944 7752 77501 77592 77684
Diameter LE Inches	20.82 21.51 22.2.13 22.2.2 23.29 24.38 25.43 25.43 26.43 27.41 28.42 30.27 30.27 31.28	Diameter LE Meters .5288 .5464 .5464 .5611 .5771 .6193 .6457 .6713 .77219 .7455 .7689
		ı

APPENDIX 5

MANUFACTURING COORDINATES FOR SECTIONS NORMAL TO STACKING LINE

TABLES IX AND X

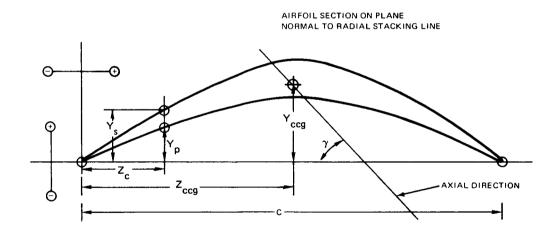


Figure 51 Airfoil Coordinate Labels for Manufacturing Sections

	٧ ٠	.0003	•0019	• 0035	•00.52	8900	-0085			0134										9770							•0231		•	.017	Ģ	000.	н	11		н	= .000259	
HETERS	<u>م</u> ۲	0003	.0009	.0020	.0031	7 4 G C .	+000 S4	4 4 4	.0376	8 4 7 7	6600	.0110	.01:20	.0130	.0139	. p10.	-0154	• 0161	•0166	1717	1111	9710 8710	-0177	.0175	.0172	.0166	.0157	.0145	.0128	.0104	•0367	-•0001	(METERS)	(METER	(METERS)	(METER	(METERS)	TERS
•	20	0000	.0025	.0.051	-0076	1010	10127	0152	2010	200	6223	.0253	. 0278	.0304	•0329	.0354	-0380	60409	-0430	90#20		9000	7550	.0582	•0608	.0633	•0658	.0684	•040•	73	•0760	.0785	RADIUS	CHORD	ZCSL	YCSL	LER CHE	TER (METE
Ω	S 30	0111	.0753	1396	201	7597	. 4441	1 6	7304	7000	5871	40.40	6903	7392	.7883	8348	.8779	9160	. 94 63	• 96 9¢	1 1 2 2 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5	9777	7.400	1881	9719	9459	. 9077	8540	.7787	ف	4955	5200	8.684	3.090	1.8001	.6574	.0102	-0014
INCHES	۷.	.0105							2006											. 6719			6992		6761	5256	185	5 705	5037	4 082	2623	• 0029	_				ES) =	
NI	ZC	יי- טטטטי-	. 1660	•	•	•	•	•	0.0000	•		7 26	196	.1961	.2957			• 5948	##69°	. 7941	.8938		1000 F	2925	3921	•	. 5915	. 1169.	. 7908	•	.9902	- 8680•		_	_			_
	X.	,	2000	9700	8700	.0071	\$6.00	.0118	-0142	•0165	•018#	.0253	1770°	.0256	0274	.0290	.0305	-0318	•0326	.0331	.0334	.0336	.0337	1000	0.000	-0321	0.00	•0291	1265	•0228	~	8	11	11	11	**	00026	= .00112
METERS	*		0003	5100.	•002B	* 000	•00 60	•0076	-0092	•0103	-0125	2410.	1210	4010	10196	.0207	.0217	-0226	.0234	•05#D	.0245	.0248	.0250	1670	7 4 7 0 0	-0235	.0223	•0207	0.184	•0150	4000	+000	(METERS)	(METERS)	(METERS)	(MFTERS)	METERS)	ניסטו
•	22		0000	# Zn n *	8 + 00.	.0072	•0036	.0120	.01#5	.0169	•0193	.0217		0200	0313	-0337	0361	.0385	6040.	.0433	.0458	.0482	•0506	2000	a n	8160°	10626	.0650	0.674	.0698	.0722	074	RADIUS	CHORD	7522	1001	_	•
4	7.5		-0127	1101.	-1902	-2799	.3700	. 4631	•5579	.6482	• 72 54	.7978	1699	9356	1.0770	1 -1428	1.2025	1.2504	1.2835	1.3038	1,3152	1.3228	1 - 32 86	125.	3.5	1.25.3	214	1.1440	7		L LC	034		2.93	1.683	9 6	= .0105	400
INCHES	¥		110	6640	.1120	-1742	. 2369	.3000	. 3639	.4283	. 4931	.5572	6,149	10.00	7671	8160	0 4 5 6 6	889	919	. 9438	.9631	16	9830	.982E	37.5	0 # C # C	4 4 0	817	7777	80.00	171	014	CARA	IV LITER	: ::	TACUTA	:	_
	ZC	1	- 0000	on .	ത	•	7	*	568	.6637	. 7585	ഗ	ກຸເ	1 - 1 4 4 6	2226	3278	.6222	.5170	.6119	1,7067	.8015	. 8963	້	80.	8	17.	,	9 6		9 6	4	້		T) COTOMO	_	_	_	

TABLE IX

INCHES D METERS	YP YS ZC YP YS	0097 .0161 .00600662 .0063 .0199 .0546 .0027 .0005 .0014	4500.	.1435 .0081 .0020	.1876 .0108 .0027	60 .2314 .0135 .0035	1 .2751 .0162 .0042	8 .3194 .0189	4 .3635 .0216 .056	•0243 •036	6900° 0250 60069	•4739 •0296 •0376	7 .5030 .0323 .0281 .012	.5298 .0350 .0687	.3595 .5544 .0377 .0091 .0141	.5783 .0404 .0095	88 .6016 .0431 .0399	.6231 .0458 .0102	.6408 .0485	5 .6528 .0512 .0105	9 •6582 •0539 •0106	47 .6555 .05E6 .0105	7 .6480 .0593 .0104	.0620 .0102	1 .6160 .0647	.5895 .0674 .0093	5 .5541 .0701 .0386 .014	.5072 .0728 .0677	53 .4449 .0755 .0065 .011	7 .36CE .0782 .0049 .009	.0809 .0328 .00	0123 .0210 .08360003 .0005	1 = 9.594 RADIUS (METERS) =	3.289 CHORD (METERS) =	1.8697 ZCSL (METERS)) = .4110 YCSL (METERS) =	= .0100 LER (METERS) =	.0099 TER (METERS) =	= .5570 X-AREA(SQ.METERS)=	.1= 40.55 GAMMA-CHORD(RAD.)= .7078
	20	03 0000 -		E	•		9.	.742	•				.						1.9100						2.5467					~ .	ا رح	ะก	RADIUS (CHORD		YCSL	17 LER	2 TER	X-AREA	ANNA-C
ME TERS	ZC YP YS	.00000003 .0003 .0026 .0007 .0016	052 .0016	079 .0025	105 -0035	131 .0044	157 •0353	183 .0061	209 •0070	. 6200 922	262 .0388	9600 - 882	314 -0103	340 .0110	367 .0116	393 -0122	119 .0127	445 •0131	171 .0134	497 -0137	524 .0138	550 .0138	576 .0138	502 •013	28 .0132	9210 - 625	9110. 180	10.	0000 0000 0000 0000 0000 0000 0000 0000 0000	759 •0074 •01	กา. ๔ กกก. ๔ ๘ ฦ	.c8120002 .00C	S (METERS) =	(METERS) =	(METERS) =	(METERS) =	METERS) =	METERS) ==	A (SO.METERS)	-CHORD(RAD.)
INCHES	YP YS	0102 .0104	637 .117	-	•	.277	072 .330	. 383	•	. 488	3454	.3774 .571	609° 020h°	-4342	*,	. 4805	• 4 995	. 5157	.5285	. 5379	-5435	80 at 1150 *	.5416	5331	.518	4366	94560	5 4 7 4 5	25679	2900	•1766	- 00 97	= 9.1	7 M	. 1.8	11	•	"	· · · · ·	DEG.1= 36.8
	22	-, 0000	. 2062	309	412	.5154	. 5185	.7216	.8246	-927	m	•13	.23	# M #	**	.54	• 64	• 75	.85	95	90.	.16	.26	37	2 - 4 739	?	89	20 (80 (86	9	•19	RADIUS (1	۵			LER (1		-AREA	GAMMA-CHORD(

																																		.2688	363	473		.000244	#97000 000000	8229
		45	.0003	•0010	.0017	•0025	.0032	•0039	•0046	•0652	•0029	• 000	•0072	6/00*	# # # # # # # # # # # # # # # # # # #	8800.	1602	#6 DD.	9600*	-0097	5600	•0100	2012	-0100	8500	Ď,		7000	4900	-0053	M	-0023	.0003	11	н	11	11	. i	•	• • • •
	METERS	4	0002	-0002	•0006	•0010	.0014	•0018	•0022	.0025	•0329	.0032	-0336	•0339	2000.	5 1 20 4 5	7 400.	6400	.0051	•00 52	.0052	•00 53	•0052	•00.51	3500*	8 500 6	400.	4 600	-0036	1000	.0017	•0008	0002	CHETERS	(METERS)	~	(METERS)	TERSI	ER (METERS)	AMMA-CHORD(RAD
	-	ZC	0000	.0028	•0026	* 008	.0112	-0140	.0167	.0195	.0223	.0251	.0279	-0307	.0335	•0363	.0391	•0419	9440	4740.	•0502	.0530	.0558	•0586	• 190	2490*	.0670	9690	67/0	1870	.0809	•0837	.0865	RADIUS	CHORD	ZCSL	YCSL	LER (METERS)	TER (METERS)	GAMMA-C
	ш																																			_	S	ю.	=	n.
Ä		۲s	6630*	•0393	.0684	0260•	.1250	.1526	•1796	.2059	•2319	.2582	-2851	.30 92	3300	.3467	.3596	• 36 97	.3774	.3835	.3882	.3923	.3942	.3918	.3839	• 36 98	3486	9178	25201	2070	1536	.0893	=	2	P)	-			-010	47.15
	v		۰	s	6	0	4	m	LO.	H	2	6	€ .	7	~	_	6	_	_	0	LI)	-	9	(.	M	ω,	- •	10 4	٠.	4 Pr		_							
CKING	INCHES	4	008	• 00 8		.0410				.0991		•125		• 1532								•			• 196						• •	031	600	(I NCHES)	INCHES)	(INCHES)	INCHESI	(INCHES)	(INCHES)	-AREA (SG. IN.) AMMA-CHORD(DEG.
TO STA		22	0000	.1098	.2197	• 3296	• 4 394	. 5493	.6591	.7690	.8788	-9887	1.0985	1.2084	1.3182	1.4281	1.5379	1.6478	1.7576	1.8675	1.9774	2.0872	2.1971	2,3069	2.4168	2,5266	2.6365	2.7463	2.8562	10000	3.1857	3.2956	3.4054	W	CHORD				TER	X-AREA (GAMMA-CH
FOR SECTIONS NORMAL TO STACKING LINE		٨٤	•0003	.0012	•0021	•0031	0400	6 400	•0058	.0067	•0075	.0084	2603	•600•	•0105	-0110	3115	0119	•0123	127	0130	•0132	.0133	•0133	• 0130	9710	310°	100	50 B	7807	2 900	90039	*000	.2551	6687	.0476	•0080	.000246	662000-	.7656
Ě			7	•	7	•	•	٠	7	•	7	•	7	ͺͺ	•	٠,	•	٠,	•	٠,	•	٠,	•	٠,	•	•	•	•	•	-	•	٠	•					+1 1		
FOR SEC	HE TER S	ď	- •0002	.0004	6000*	.0015	•00 21	•0026	•0032	.0037	-0042	7 # CD*	• 0053	•0358	2900	•0366	0700.	•0073	50015	1100	8 / 00 .	.0078	8700.	1100.	100.	200	1000		-0052	000	.0032	.0017	00003	(METERS)	I ME I ERS	(METERS)	(METERS)	ET ERS)	THE LERS!	CHORD(RAD.)
		ZC		.0028	•0055	.0083	•0110	.0138	•0165	.0193	•0221	•0248	•0276	-0303	.0331	•0358	.0386	# C # C # C	I b b D ·	6940.	9640.	•0524	.0551	•0579	1390	40404	2000.		4470	6777	.0800	.0827	• 0855	RADIUS	CHORD	ZCSL		x 3	V-ADEA	GAM MA-
	Ш																																			ın I	0	ا ا	ے م	+
		YS	•0101	.0471	.0838	.1201	.1560	.1916	.2270	.2624	-2970	#0K	.3622	3908	-4142	6 £ 2 3 3 8	.4518	* 4686	2 4 6 5 4 5 5 6 5 6 5 6 5 6 5 6 5 6 5 6 5	466h•	5111	.5195	.5251	. 5234	5119		44/4		3706	715	.2478	155	18	10.04	5.56	1.87	.31	900	70.	43.86
	S		60	2	2	9	พ	32	Š	55	m	2	75	<u>ق</u>	a (<u>.</u>	2	60	.	2	&	<u></u>	M)	<u>ص</u> ا	P) (<u>.</u>	- ų	n 6	מי ת									11 1	_	
	INCHES	¥	00 0	5 .01	1 .03	9 .05	2 .08	7 .10	.12	•14	• 16	. 18	•20	. 22	.24	• 26	5 .274	. 28	•	. W	•30	. 30	•30	25	•	87.	•		206		.125	90.	01	INCHES	HES	(INCHES)	X¥.	3.H.S	INCHES	ORD(D
		ZC	·	H	_	32	~,	5	65	75		6.	õ	ij	5	7	5	• 62	5	60	91	90		. 27	80 S	5	9	- (2228-2) -	25	35	ADIUS	HO 20	SCSL	CSL	ا عه	E K E K E K	AMMA-C

TABLE IX

TABLE IX

J ETERS	ZC YP YS	*0000 - *0005	9000 -0000 -0000	.0060 .0001		7000 0000	Inn. Funn. uzin.	100. 4000. 0010.	.0180 .0005 .002	.0210 .0606 .062		.0270 .0006	.0300 .0007	.0331 .0007	.0361	8000. 1620.	.0421 .0008	•000° 15h0°	.0481 .0510	.0511	.0541 .0010	0100. 1000	1100 .EJG	0100 T000	000- 1690-	.0721 .0008	-0751 -0007	.0781 .0006	4 .0811 .0004	3 .0841 .0003	5 .0871 .0002	nn•	28 •0931 •0002 •0001	RADIUS (METERS) =	CHORD (METERS) =	Z ZCSL (METERS)	YCSL (METERS) =	LER (METERS) = .00021	TER (METERS)	
INCHES	YP YS	0081 .0081	•0015	#		n r	3 0	73	6 0	a n	m	10	u	m	ĸ				.0376 .219		.0398 .2393			1077° 4750°		1620	.0260 .1435		_	18 .0	.0060 .0	sn. 20	-•0069 •002	1	M	-	"	"	"	•
	2.0		1183	2366	0002	n (*)	• 4 / 32	-5915	.7097	.8280	.9463	1.0646	1,1829	1.3012	1,4195	1.5378	1,6561	1.7744	1.8927	2,0109	2.1292	2.2475	2.3658	7-67-5	2,7207	2.8390	2,9573	3.0756	3,1939	3.3121					CHORD (IN	_	_	LER (IN		2
	YS	2000	7000		TT0.1•	•0015	.0018	•0022	• CC25	AC00-	200	-003E	0400	*****	8 400	•0052	.0055	•0029	2900*	•0065	•0066	9900	*900°	2400	.0051	-0046	.0041	•0035	• 6 2 3 3 •	•0023	• 0017	•000 •	* C C C C		11	11	11	11		
METERS	4	2000-			7000*	•		•		. '						•	•	•	Ī		•	.0014	.0314	. 0013	1100	.0010	5000°	.0007	.0005	*000	.0002	0001	0003	(METERS)	(METERS	(METERS)	(METERS	ET ERS)	(METERS)	27
_	20	C	3 6	0.000	0900.	•0000	.0120	-0150	0180	8020	970	6320	6620	P 2 2 3	0359	.0389	.0419	6440	e2#0*	•020•	•0539	.0568	•0598	9790.	8890*	.0718	8 44 0 .	•0778	. D8C8	.0838	36	m	32	RADIUS	CHORD	ZCSL	YCSL	LER (ME	TER (ME	į
	۲s	Č	0000	90700	.0421	•0576	.0719	084	90	900	, и) r	-	1735		-2035	2117	.2315	#	-	•2608	.2609	.2513	2.567	2013	.1813	29	38	w	•0925	67	37	.0073	12 • E	200	1.977	102	- 0085	900	•
S	4	6	•	7000	6/00*	•	•	•	•	•	2440	•	•	•	• 1	6 3 0		.053	•05	• 056	.05	•05	.05	.05	- C - C - C - C - C - C - C - C - C - C	.03	_	.027	•	. 014	•006	-, 002	1		_	: ::				
INCHE			٠ د									_		٠.,		٠.		60	ω	m	-	a n	~	9	2 -	268	40	#	N	C 1	600	'n	M	···			: :		-	

TABLE IX

	۸s	*0005	• 0005	9000•	0100	•0012	600	•0015	.0016	•0019	•0C22	•0026	•0029	•0032	•0036	• 003 9	-0042	• 0045	7400	.0050	•0051	.0051	5400	. 0045	0 4 0 0 •	5003	52023	* 700°	50013			5	· cno ·		0460* =						* CC024
NE TERS	d b	- •0002	0001	- •0000	2022*	*0000	0000	0000	0000-	- 00001	0001	1000-	- •0001	0001	- •0001	0000	•0001	.0002	•0003	.0003	•0003	*000	*000*	• 000	*000*	.0003	5000	7000	2000-	֓֞֜֜֜֜֜֜֜֜֜֜֜֜֜֜֜֜֜֜֓֓֓֜֜֜֜֜֜֜֓֓֓֓֜֜֜֜֜֓֓֓֓	220.	חחחי	-• ann -	(METERS)	(METERS)	(METERS)	(METERS)	5	1000	15 K 57	X-AREA (SG.METERS)
	2C	0000	.0030	.0061	.0091	•0121	0152	-0182	-0212	00242	-0273	.0303	.0333	.0364	•039#	*C#5#	•0455	.0485	•0515	•0546	•0576	9090	•0636	.0667	-0697	.0727	86/00	99/0	8 1 8 0	0 0 0	n (0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	5050	3 460 •	RADIUS	CHORD	1532	1507	1207 1207 1	TER CHELERS	ובא יחב	X-AREA (
_	۲S	• 00 80	.0194	•0298	.0391	.0471	15.7 B	±05.87	. הפנים	-0747	20872	.1008	.1145	.1278	-1406	.1529	.1648	.1763	•1862	.1949	•2010	.2022	-1947	.1777	.1574	.1361	-1147	243	•0752	2000	- 0 4C4	•0236	8 400	13.764	m	1.9652	ָבְיבָיבְיבָיבְיבָיבְיבָיבְיבָיבְיבָיבְיבָיבְיבָיבְיבָיבְיבָיבְיבָיבְיבְיבָיבְיבָיבְיבָיבְיבָיבְיבָיבְיבָיבְיב	7000	7900	0000	.3822
INCHES	Ϋ́	0079	P#00*-	0016	.0003	. 0013	4100	0000	8000	- 0025			m		0020			a			35		en i	-0147	.0141	-0131			o (,	1000	·	0081	(INCHES) =	(8				(TNCHES)	_ '	(See IN.)
	2 C	-, 0000	.1193	.2386	3580	4773	7 7 7 7	7150	2 4 4	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	1 0739	1.1933	1,3126	1.4319	1,5512	1.6706	1,7899	1.9092	2.0286	2.1479	2.2672	2.3865	2.5059	2.6252	2.7445	2.8638	2.9832	3.1025	3.2218	2145.5	3 - 4 605	3.5798	3.6992	RADIUS (IN	~	_	•			רא ה	X-AREA (SG
	¥.S	2000	1000	6000	8000	•0010	•0013	•001#	•0016	•0018	•0020	•0024	7702.	•0031	4200	•0041 •0041	4400	7.0047	6 too.	•0051	.0053	•0052	•0020	•0045	0400	•0035	• 002 9	•0024	• 0019	•0015	• 0011	8	2002•	88 48	0 0		1000		= •C00208	= • ccoz	= +000251
HETERS	\$	6000	7000	1000-	0000-	0000			.0001	•0001	0000	0000		חחחחי -																	0000	- •0001		CMETERS			<u>ر</u> د	ERS)			EA(SO.METERS):
	20	6	3000	0500	Tanne	•0031	.0122	-0152	.0182	•0213	.0243	-0274	+050.	40354	90303	00000	, C	98 # 0*	.0517	-0547	.0577	.0608	.0638	•0669	• 0699	•0729	.0760	•0790	•0821	82	80	g	.0942	PADIIIS	משטאט	7007	100	Yest	LER THET	TER (MET	X-AREA (S
¥	\$	6	0000	10201	.0312	-0411	.0497	•0568	.0627	•66 90•	• 0,805	046D*	• 1079	1218	7007	1505	1727	1347	1941	.2021	.2071	•2066	.1961	.1780	.1577	.1367	.1156	*095¢	•0765	∞ ₁	≂	28	0800	13.614	70.2		716.	.0603	-0082	5	.3897
INCHES	4		6/00°-	0039	0007	.0017	•0032	.0038	•0036	.0026	•0014	. 0002	•	- 0004	9000	8700		3110	•0139	.0155	.0165	. 01 70	.0170	.0165	-0154	.0139	.0119	• 00 95	•0066	. 0032	0005	*	₩600°-	TACHES		, .	٠.	_	INCHES) =	£ 2)	11 (NI '05
		9	222	1196	393	589	4 786	382	7179	8375	•	16	.1965	9 ;	0	6754		144	2-0340							9716	912	2	a	0	ത	895	1602	ADTIIS CTN	•		7		H	_	EA LAG

	S ★	•0002	* 000a*	*000J	9000	0100	-0012	-0012	•0013	\$100.	•0017	0200	•0024	.0027	•0031	•0034	•0038	.0041	* * * * * * * * * * * * * * * * * * * *	•0046	6400-	1500.	•0020	7 4 2 2 4 2 2 4 2 2 4 2 2 4 2 2 4 2 2 4 2 2 4 2 2 4 2 2 4 2 2 4 2 2 2 4 2 2 2 4 2	2400*	•0036	35.55	-0023	9100	0100	•0006	2000-			11	11	11	•	11	1= 1.0637
FE TERS	٩×	- •0002	5001	0001		- 7001	0001		0002	-,0003	- •0003	0003	- •0003	0003	- •0002	0001	2000	2000.	•0001	.0002	-0002	• 0003	•0003	.0003	enno.	•0003	7000	*000		1000	0001	0002	(METERS)	(METERS)	(METERS)	(METERS)	(METERS)	(METERS)	X-AREA (SO.METERS)	GAM MA-CHORD (RAD.)
	ZC	0000	.0031	,00.62	1600	0124	7110	0.100	-0217	0.248	0278	0303	•0340	.0371	-0402	.0433	*9*0*	.0495	•0526	.0557	.0588	.0619	•0650	.0681	2110	-0743	*//0"	208D.	1000	0897	0.928	.0959	RADIUS	CHORD	ZCSL	X CCL	LER CHE		X-AREA	GAM MA-C
Z	YS	9700	6177	.0265	1440	1 0 0 0	7070	6040	2150	0571	-1986	7670	-0937	1081	.1219	.1353	•1482	.1607	.1721	.1824	•1919	. 1 989	•1969	.1852	.1651	-1421	.1189	7960	00.00	2/50.	2000	.0067	4	3.77	~				.378	20
INCHES	d.	0079	200	2000	֓֞֜֜֜֜֜֜֜֜֜֜֜֜֜֜֜֜֜֜֜֜֓֓֓֓֜֜֜֜֜֜֜֜֜֜֓֓֓֜֜֜֜		77000	1000-	1001	1010	4210-1	0136	-, 0135	0119	- 00 92	0057	0017	.0018	. 0048	.0071	6800°	.0102	.0109	.0111	•0108	.0100	•008E	.0072	7670.	0000		+800*-	I CAUCATO	, C			INCHES !		_	. 🙃
	2 C	ר מחמח	•	01710	1647.		0.00	16091	01010	9700	145C	1.2183	1 3401	1.4619	1,5838	1.7056	1.8274	1.9492	2.0711	2.1929	2,3147	2.4366	2.5584	2.6802	2.8020	2,9239	3.0457	3,1675	3.2894	3.4112	3000) ~	0110	-	_	•			_	ပ
	45		7000	5000	/ 00.0*	0110	1100	-0013	+135	9700	01010	1200	1005	.0031	.034	•0038	.0041	****	• 0046	6400•	• C05C	-0051	6 400	•0045	2400	•0035	5 200	*DDZ#	6707	0110	500	.0001		96U*	11	11	111	_	± .00024	= 1.0246
METERS	<u>a</u>	0	Z000*-	1000-		0000-	0000		י מממת	1000-	1000-	2000	1000	0002	0001	0000-	.0001	.0001	.0002	.0003	.0003	•0003	*000*	*000*	*000*	•0003	• 0003	.0003	2000.	1000		0002	CMCTCDC	(METERS)	METERS	CMETERS	(となり上しま)	MFT FRS 1	EA (SO . METERS)	CH ORD (RAD.)
Σ	2C		0000	•0030	1 90 0	1600	.0121	.0152	2910*	2120.	.0243	2777	00000	4660.	7950	-0425	.0455	.0486	.0516	•0546	.0577	.0607	.0637	•0668		.0728		0789		2685.	n.		0,1	CHORD	1002	4 C 2 K	103F	TFR (187	X-AREA(GAMMA-C
	45		0800.	•0189	•0289	-0377	.0452	┥	.0558	ا ف	98	H 1	9660	1001	1 2 5 7	11481	1602	.1719	1821	.1913	.1985	.2016	.1948	.1790	.1592	.1375	.1157	0360	•0756	0.57	_ i	.0052	,	13.314	•	4	•	•	74747	= 58.71
INCHES	٨b		0080	0047	0022	0005	*000	-, 0001	0012	0030	00 51	0071	0083	0084	1700	1 0017	֓֓֓֓֓֓֓֓֓֓֓֓֓֓֓֓֓֓֓֓֓֓֓֓֓֓֓֓֓֓֓֓֓֓֓֓֓	. 00 55	4800	. 0107	.0124	013	.0143	.0145	014	013	.0118	6600*	0	5	90.	-,0080		CHES)					-	. 7
	22		0000	.1195	.2390	. 3584	.4779	.5974	.7169	36	. 955	075	ᅻ '	•	•	ຄຸບ	1.7927	•	ָר בּי	2.1506	270		503	628	. 748	98	.987	.106	3,2259	345	464	3.5844		SOL		1.	YCSL (1		4	- 판

NCHES	0		METERS			INCHES		<u>م</u>	METERS	
	YS	ZC	\$	S	20	۸k	x x	2C	۸۶	YS
00 80	•00.80	0000	- •0002	•0005	0000	7.000-	.0077	0000	0002	2000*
0100	0130	.0033		.0003	37	0134	• 00 92	•0035	- •0003	*0005
0128	•01.70	0	- •0003	#000 *	•2759	0196	.0102	0.000	0005	.000
0165	0200	S)	+000-	\$000	.4138	0265	•0106	•0105	0007	•0003
0212	021	•0131	- •0005	•0000	.5518	0339	.0103	.0140	0009	• 0003
26.7	.0225	.0164	0007	9000	.6897	0419	• 00 95	•0175	- •0011	•0005
0330	022	.0196	- •0008	9000	.8277	0503	.0080	.0210	0013	.005
402	0.0201	.0229	+	\$000	ຸດາ	0592	• 0029	€ 0245	- •0315	•0001
0479	- B183	.0262		2000	1,1036	0691	.0031	.0280	0018	.0001
8 × 5 C - 1	66.0	20295	·	•0005	1.2415	0784	.0010	.0315	0020	2020*
	.0257	0327	1	2000	1.3795	0829	•0031	•0320	- •0021	.0001
1055	03.55	-0350	001	6000	1.5174	0828	0100	.0385	0021	.0003
	6240	0393	•	20012	1,6554	0811	.0229	.0420	- •0021	•0006
•	.0511	20404	·	•0016	1.7933	2620-	0.378	.0456	0020	.0010
7 0 5 0	0470	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	1	6190	1.9312	0775	.0534	.0491	- •0020	-0014
- 0493	#980°	.0491	,	•0022	2,0692	0759	.0685	.0526	0019	.0017
_	.0983	.0524		• 0025	2,2071	0742	.0832	.0561	- •0019	•0021
•	•10 98	•0556		•0028	2.3451	0724	9760.	9650.	0018	.0025
76	.1209	.0589		.0031	2.4830	0765	.1113	.0631	0018	.0028
0472	.1307	•0622	- •0012	•0033	2,6210	0683	.1217	9990*	- •0017	•0031
0467	.1372	•0655		•0035	2.7589	_•C66C	.1266	.0701	0017	.0032
.0461	.1368	.0687		•0035	2,8969	0633	.1223	•0736	0016	.0031
.0453	•1259	•0720		•0032	3.0348	0603	.1088	.0771	0015	•C028
1940	.1067	.0753	•	•0027	3.1728	0569	*092¢	9080°	0014	-0023
.0425	•0852	•0786	ı	•0022	3.3107	0531	.0762	.0841	0014	.0019
403	• 0643	a n	001	•0016	3.4487	0485	•0598	.0876	0012	•0015
.0374	•0468	∞ .	- •0009	•0012	3.5866	0432	9440	.0911	0011	.001
-0332	34	.0884	0008	6 000	3,7246	0372	.0324	9460*	- •0009	•0008
277	.0249	.0916	0	9000•	3,8625	0304	•0242	.0981	- •0008	•0006
213	8	ത	- •0005	•0002	# 0000 #	0229	.0189	.1016	0006	• C 00 5
	.0131	9	0004	• 0003	4.1384	0146	•01#	.1051	+0000-	*00 0 *
21	•0020	-	- •0001	•0001	4-2763	0055	.006	.1086	0001	-0002
22	-	RADIUS	(METERS)	11	RADIUS (I	I NCHES)	= 16.044	RADIUS	(METERS)	
INCHES	3.99	CHORD	(METERS)		_	_	4.27	CHORD	_	-1086
		ZCSL	(METERS)	11	_	_	7	ZCSL		
INCHES) =	000	YCSL	(METERS)	н	~		•	YCAL	(METERS)	11
	.008	I		•	. ~			LFR	(MFT FRS)	H
	00.	Ξ	(METERS)						(METERS)	
-	7.7				•		,			
	7	Ł	EA (SO METERS)	•	400	•		0	FALS METERCI	•

ROTOR MANUFACTURING COORDINATES

		۲S	*0005	1000	00000		0002	0003	+000°-	0005	2000		0001	.0004	•0008	.0013	-0017	.0021	9230.		0000	•0026	.0022	•0019	ATOO.	00100	•0008	.000	9000*	9000*	H	11 1	11 I	(}	ı II	11	.)= 1.2
	ME TER S	Q.	0002	0005	- 10008	7100-	0016	0018	0021	- •0023	-• UUZ 6	5200-1	3500	0029	- •0029	0028	0028	0027	- 00026	4600	0023	0021	0020	0018	9130	- 00113	0010	8 3 3 3 ° -	- •0005	0005	(METERS)	(METERS)	X	METERS	(METERS)	X-AREALSO . METERS)	GAMMA-CHORD (RAD.)
	œ	ZC	0000	.0038	.007		.0188	•0226	.0263	.0301	. 0358	9100	1045	6840	•0526	*0564	*0602	.0639	.0677	6360	2670	0827	.0865	•0902	0 4 4 0	1015	.1053	.1090	.1128	1166	RADIUS	CHORD	ZCSL		LER CHE	X-AREA	GAMMA-C
Ä		۲s	-0077	.0042	8000	0700	6800*-	0120	0150	0183	0212	- 10203	7200	10144	•0320	4640.	•0662	•0829	.1012	1511.	1151	1015	.0872	.0751	.0634	£150.	0319	.0261	.0233	.0221	-	•		1	.0079		72.44
DINATES KING LIN	INCHES	ΥP	0077	0188	~ :	* 0				2		- 114		-1156	-,1135	1111	-, 1085	1055	1021	0.000	2000-1		0782	0719	6 490 -	0573	7.040.7	0306	0202	0092		_	INCHES) =	_	(INCHES) =	_	~
ING COOR L TO STAC		22	0000	-1480	.2961	* 4	.5921	. 9882	1.0362	1.1842	1-3323	1.4803	1.775	107751	2.0724	2.2205	2,3685	2.5165	2.5645	371847	2.3605	3.2557	3.4047	3,5527	3.7008	80 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	0 4 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5	4.2929	6044.4	4.5889	10	_	_	_	LER	X-AREA (S	GAMMA-CHORDIDEG
KOTOK MANUFACTURING COORDINATES FOR SECTIONS NORMAL TO STACKING LINE		YS	2000	-0002	2000-	7000	-0001	.0001	2000•	0001	0001	1000-		•000	.0012	9100*	•0020	# ZDJ*	*0028	12 CC	.0000	•0027	.0023	.0019	• CDT 6	5 DOJ •	1000-	9000.	•0002	•0003			11 1	; ; ;	= .000211	182000 = (5	
FOR SE	HETERS	Ϋ́Р	-,0002	#000° -	9000		0012	0014	0017	0019	0022	0023	0023	0023	0023	0022	0022	0021	- • 00 ZD	9100	0018	7100	0016	0015	0013	0010	0008	0006	*000°-	- •0001	(METERS)	2 ((METERS)	נאם נישט ניאס ניא	(METERS)	-AREA (SG.METERS)	GAM MA-CHORD(RAD.)
		22	0000	•0036	.0072	2010	0179	.0215	.0251	.0287	-0323	.0359	0.00	9940*	.0502	.0538	.0574	.0610	.0645	7470	.0753	.0789	.0825	-0861	.0896	8960	4001.	.1040	.1076	.1112	RADIUS	CHORD	ZCZT	:	TER (ME	X-AREA	GAM MA-C
	O	۲۶	9200*	•0077	*00.	8900	7 5000	002	000	~	•002	0035	0.15	.0322	048	064	.0801	.0961	11097	0011.	18	1058	.0911	16	4٠	40.0	2.7	-4	.0185	.0103	16.2	4.376	2.379	070		450	71.0
	INCHES	4.6	0076	=	220.	. 050	0367	0.56	.065	0.75	857	1921	1,0932	9 0		0	-• 0820	0829	٦, c	֓֞֜֜֜֓֓֓֓֜֓֜֓֓֓֓֓֓֓֓֓֓֓֓֓֓֓֡֓֓֓֡֓֓֓֡֓֓֡֓֓֡	0716		90.	50	50.5	9 6 0	032	024	0	0.2	N CHES)	NCHES)	is i	~ .	NCHES) II	IN.	DE 6.1
		zc	0000	.1412	.2823	4239	. 5647	8470	.9882	•	•					•	•	٠	•		2.8233	•		•			•		•	•	RADIUS (I)	_	_		TER CINC	-	GAMMA-CHORDE

S B PETENS	75 YP YS	• 1500•- 0000• /550•		0000 0000 0000 0000 0000 0000 0000 0000 0000		.1508 .0025 .	, 1600 8900 E841.	. 2150 . 0118	. 2459 . CLD 6245.	.2759 .0157 .0047	.3041 .0176 .0052	.3295 .0196 .0057	.3516 .0216 .0061	,3706 •0235 •006s	. ჰგ64 . ტ255 . ტ66 გ	. 3992 .0274 .0071	. 6290 .6294 .0073	.4158 .r314 .co74	. 4197 . 65333 . 6074.	1 .4205 .0353 .0076	1 .4178 .C372 .CG74	4116 .0392 .007n	4518 .6412 .5673	3481 .C+31 .0070	3703 .0451 .0667	. 3481 .0470 .0663	.3211 .0490 .0052	3 ,2886 ,0510 ,0052	3 .2500 .0529 .0044	3 .2043 .0549 .0036	3 .1502 .0568 .0025	6 .0853 .0588 .0613	6 .0102 .06070601	1 .0067 .06080001	10.7 s pants (wetfor) = -272	0.40 - 1.00 MARCA	0000 1 (02014E) 02007 7600 -	1500 # (CKH) HW) 1507 0567*1	*2806 YCSL (METERS) = +00/1	0053 RLE	STOOPS IN (SEEERS) IN STREET STOOPS	I = .2077 X-AREA(SQ-METFRS. = .00013	*)# 23.26 GAMMA-CHORD(RAD.)# +406
I NCHE	4 y	33 000	800°- 8400°	70. 7// Hari	יים. קיים שור		3858	4630	5402 +16	6173 .18	645 .20	717 .22	488 .24	9260 .25	032 .26	.0803 .27	.1575 .28	2347 .29	.3118 .29	.3890 .29	4662 .29	.5433 .29	• \$202 • \$9	6977 .27	7748 •25	.8520 .24	.9292 .23	.0064 .20	.0835	.1607 .1	.2379 .0	3150 .	.388700	.392200	PHU. (AUTOM				SL LINCHE	STE CINCHES	TE CINCHE	-AREA (54. 1N	AMMA-CHORD (D
v	٧\$	1, .0001	99.	000				· ·		,	•	٠.		. 0	•	3.	0• 9	٠	٠.	٥	•	•	٠.	,	•	•	ا ب		•	7	~		_		R 1 = +264	090• = (54	c5) = •031	830 = 1Sc		47 - CEC - B		7 3 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7	19F. N
METER	ZC YP	0000- 0000	90	00. 8000			00. 2650	0116 .00	136 .00	155 .00	0174 •00	194	0213 .00	00.	252 .00	271 .00	0390 .00	310 .00	329 .06	348 .00	368 .00	3 987 .0C	00. 2545	07.0	30.	00. Setu	00. 1870		C523 .0G	00. 2.40	20			00.0	ILS (MET	134) UNE 1	(P. F. T	2.	TEOS.	1 m		35.00.00.00.00	ቲ 5 1
нЕѕ А	S >		0.96	****	111. 012	745	243	510 .220	778 .25	046 ,29	312 :32	554 .35	770 .38	4. 656	122 .42	257 .44	367 .45	446 344	500 .44	524	518 .46	482 .46	611	fr. 607	101	976	750		7. 67.	7.007	7	, , 0 (760	5) = 10.41	s) = 2,	5) = 1,240	5) = .31b	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	٠,			02 + 1 • 5
INCHES	Zr Y	0 0000	50 - 04	1524 -0	287	3049	3611 .1	4573 .1	336 .1	098 •2	60 .2	622 .2	384 .2	47 .2	6066	671 .3	.1433 .3	.2176 .3	.2458	.3720 .3	4482 .3	.5244	F. 200		E • 1657	Z • E 42 H •	3.000	2. HIB.	2.	1. 2761.	** F017*	0 · /0 · /0 · /0 · /0 · /0 · /0 · /0 ·			S (TECHE	D CINCHE	JIJNI)	CHUCHE CINCHE	11.0	. I	1 · (1/2) · 1/2		

TABLE X

				STATOR FOR SECT	STATOR MANUFACTURING COORDINATES FOR SECTIONS NORMAL TO STACKING LINE	RING COOI L TO STAC	RDINATE KING LIP	SS AE			
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TABLE X

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APPENDIX 6

NOMENCLATURE

Symbol	Definition
A	areas
A/A*	(area)/(sonic flow area)
a	= distance along chord line to maximum camber point from leading edge
b	rotor semi-chord at 75 percent of span from root
c	aerodynamic chord, i.e., along the flow surface
D	diffusion factor for rotor = $1 - \frac{V'_2}{V'_1} + \frac{r_2 V_{\theta 2} - r_1 V_{\theta 1}}{(r_1 + r_2) \sigma V'_1}$
	for stator = $1 - \frac{V_4}{V_3} + \frac{r_3 V_{\theta 3} - r_4 V_{\theta 4}}{(r_3 + r_4) \sigma V_3}$
DCA	double-circular-arc
E	excitations per rotor revolution
Н	stagnation enthalpy
i _m	incidence angle between inlet air direction and line tangent to blade mean camber line at leading edge, degrees
i _{ss}	incidence angle between inlet air direction and line tangent to blade suction surface at leading edge, degrees
ĸ	blockage factor, actual/effective flow area
К ₁₋₈	radial spring rates
K _t	stress concentration factor
LE	leading edge
M	Mach number
MCA	multiple-circular-arc blade

Symbol	Definition
N	rotor speed, rpm
p	pressure
P/A	centrifugal pull stress
PC	precompression blade
r	radius
R	distance along conical surface from apex to blade (see Figure 50)
R_c	streamline radius of curvature
s	blade spacing
T	temperature
t	blade maximum thickness
TE	trailing edge
T ₁₋₄	torsional spring rates
U	rotor tangential speed
V	air velocity
W	weight flow
WA	leading edge wedge angle
x conical	distance in unwrapped conical plane
Y_p	airfoil coordinate of pressure surface normal to chord line
Y_s	airfoil coordinate of suction surface normal to chord line
Y_{ccg}	vertical distrance to airfoil center of gravity from chord line
у	length along calculation station
y conical	distance normal to x conical

Symbol	Definition
z	axial distance
Z* ratio	shroud modulus/airfoil modulus
z_c	airfoil coordinate parallel to chord line
Z_{ccg}	horizontal distance to airfoil center of gravity from leading edge along chord line
β	absolute air angle = COT^{-1} (Vm/V _{θ})
β'	relative air angle = $COT^{-1} \frac{(Vm)}{(V'_{\theta})}$
β*	metal angle, angle between tangent to mean camber line and meridional direction
γ	blade chord angle, angle between chord and axial direction
δ°	deviation angle - exit air angle minus metal angle at trailing edge
ϵ	angle between tangent to streamline projected on meridional plane and axial direction
₹	cone angle = TAN^{-1} $\frac{(r_{te} - r_{le})}{(Z_{te} - Z_{le})}$
$\eta_{ m ad}$	adiabatic efficiency
heta	circumferential direction
λ	angle of calculation station measured from axial direction
ρ	density
Σ	angle on conical surface of revolution - see Figure 50
σ	solidity or stress
φ	camber angle, difference between blade angles at leading and trailing edges on conical surface (Figure 50)
$\phi_{oldsymbol{\Sigma}}$	camber angle, difference between blade angles at leading and trailing edges on the unwrapped conical surface (Figure 50)

Symbol	Definition
$\phi_{ ext{f}\Sigma}$	front camber angle, difference between blade angles at leading edge and MCA transition point on the unwrapped conical surface
ω	angular velocity
ω_{t}	torsional frequency
$\overline{\omega}$	total pressure loss coefficient, mass average defect in relative total pressure divided by difference between inlet stagnation and static pressures
	Subscripts
av	average
f	front
le	leading edge
m	meridional direction (r - z plane)
p	profile
r	radial direction
ss	suction surface
t	total or stagnation
te	trailing edge
z	axial direction
heta	circumferential
1	station into rotor along leading edge
2	station out of rotor along trailing edge
3	station into stator along leading edge
4	station out of stator along trailing edge

superscripts

relative to roto

designates blade metal angle

degrees of arc or temperature

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